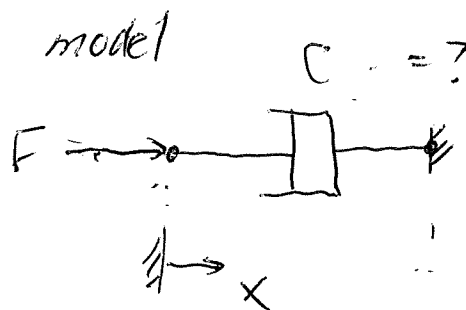
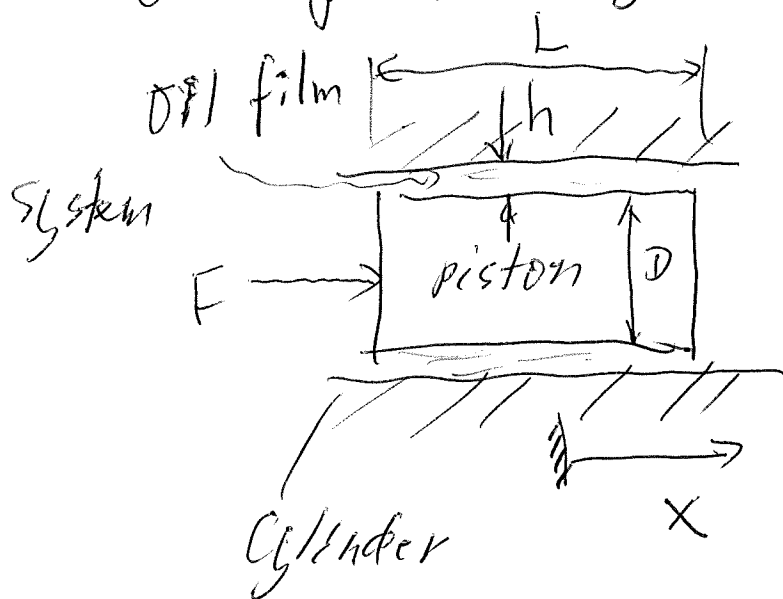


Determine C :

Example : Engine Piston-Cylinder contact



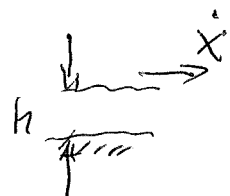
model $\rightarrow F = C \dot{x}$ (A)

system $\rightarrow F = C \dot{x}$ (B)

work out Eq. (B)

$$F = \sum \tau dA = \int_A \tau dA = \tau A = (\eta \dot{\gamma}) (\pi DL)$$

$$= \eta \frac{\dot{x}}{h} \pi DL = \left(\frac{\eta \pi DL}{h} \right) \dot{x} \quad (B)$$

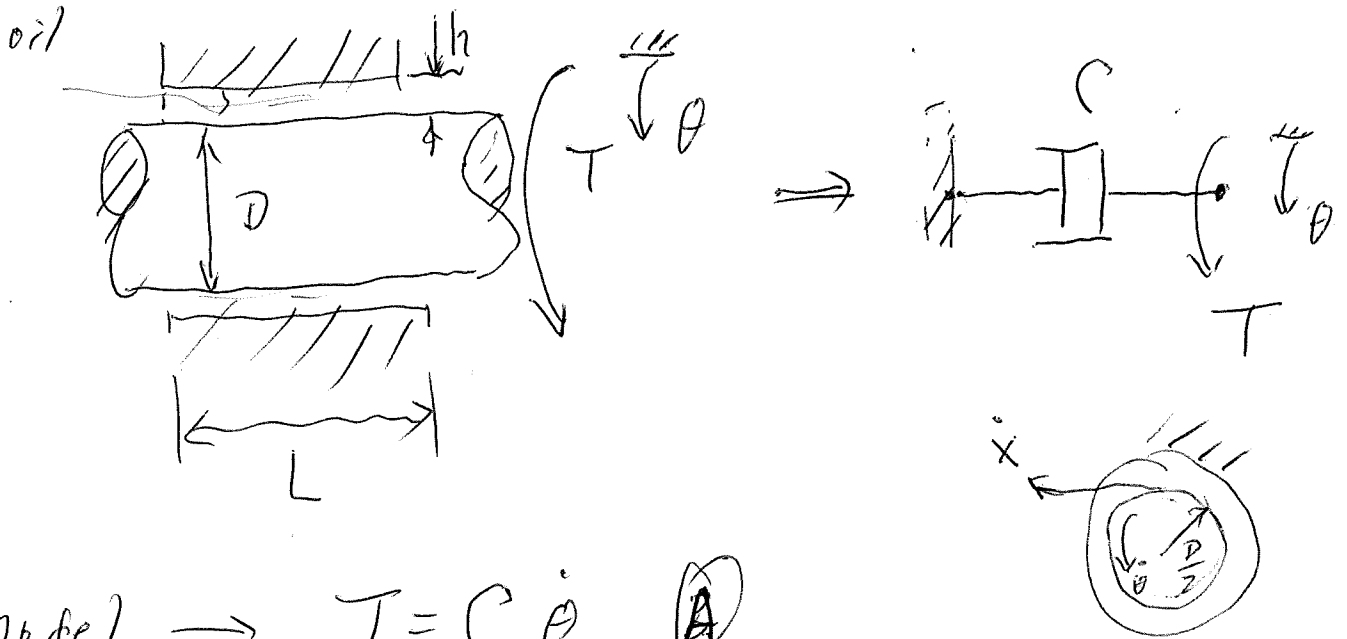


$$C = \frac{\pi DL \eta}{h}$$

$$\dot{\gamma} = \frac{dx}{dy} = \frac{\dot{x}}{h}$$

h is very small gap

Example: Engine Crank bearing



Model $\rightarrow T = C \cdot \dot{\theta}$ (A)

System $\rightarrow T = \int_A (\tau dA) \frac{D}{2} = \tau \frac{D}{2} \int dA$

$$= \eta \dot{\gamma} \frac{D}{2} A = \eta \frac{\frac{D \dot{\theta}}{2/h}}{\frac{D}{2}} \frac{D}{2} \pi D L$$

$$= \frac{\pi D^3 L \eta}{4h} \dot{\theta} \quad (B)$$

$$\dot{x} = \frac{D}{2} \dot{\theta}$$

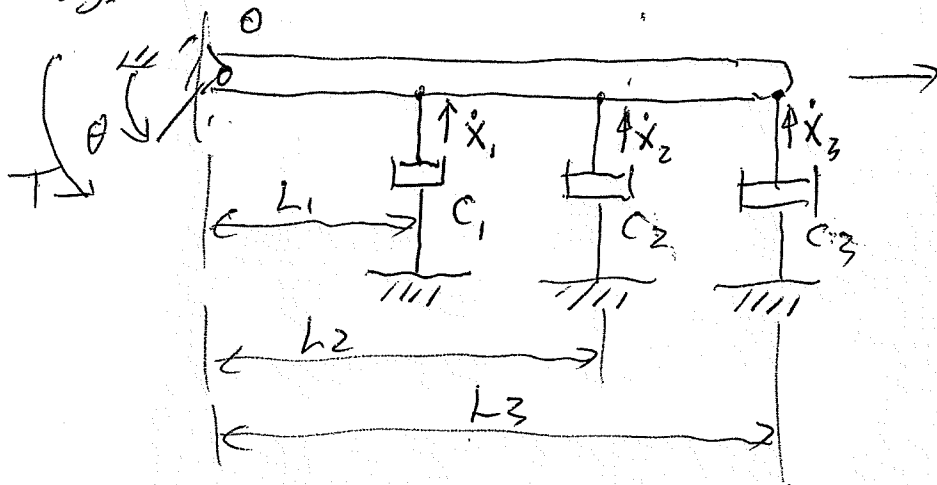
$$\therefore \boxed{C = \frac{\pi D^3 L \eta}{4h}}$$

check units

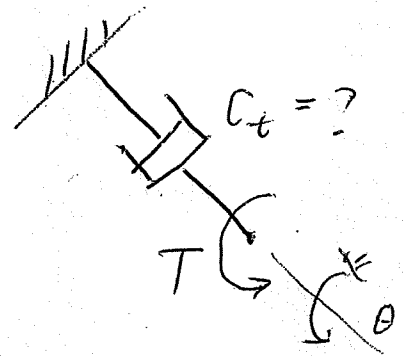
c) Other combinations

Example:

System



① Determine an equiv. torsional damper at "0" mode 1



model: $T = C_t \dot{\theta}$ (A)

system: $T = (\quad) \dot{\theta}$ (B)

$\dot{x} = L \dot{\theta}$

work out Eq. (B)

$$\Rightarrow T = L_1 F_{C_1} + L_2 F_{C_2} + L_3 F_{C_3}$$

$$= L_1 C_1 \dot{x}_1 + L_2 C_2 \dot{x}_2 + L_3 C_3 \dot{x}_3$$

$$= L_1 C_1 (L_1 \dot{\theta}) + L_2 C_2 (L_2 \dot{\theta}) + L_3 C_3 (L_3 \dot{\theta})$$

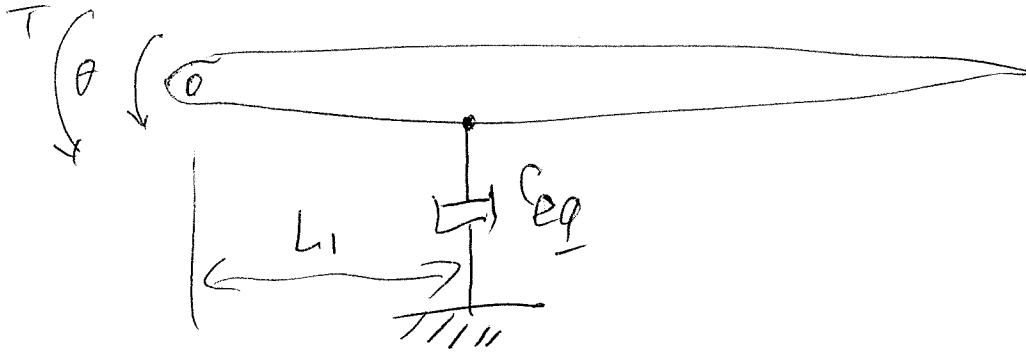
$$= (C_1 L_1^2 + C_2 L_2^2 + C_3 L_3^2) \dot{\theta}$$

$\underbrace{\hspace{10em}}_{C_t}$

translation

- ② Determine an equivalent damper at location c,

Model



Model: $F_c = c_{eq} \dot{x}_1 \Rightarrow F_c L_1 = (c_{eq} L_1 \dot{\theta}) L_1$

$$\Rightarrow T = (c_{eq} L_1^2) \dot{\theta} \quad (A)$$

System: $T = (\quad) \dot{\theta} \quad (B)$

$$T = L_1 F_{c1} + L_2 F_{c2} + L_3 F_{c3}$$

:

$$= (c_1 L_1^2 + c_2 L_2^2 + c_3 L_3^2) \dot{\theta} \quad (B)$$

$$Eq. (A) = Eq. (B) \Rightarrow \left[c_{eq} = c_1 + c_2 \left(\frac{L_2}{L_1} \right)^2 + c_3 \left(\frac{L_3}{L_1} \right)^2 \right]$$

Energy method

$$\dot{E}_d \text{ in system} = \dot{E}_d \text{ in model}$$

(1) model: $\dot{E}_d = C_t \dot{\theta}^2$

system:
$$\begin{aligned}\dot{E}_d &= C_1 \dot{x}_1^2 + C_2 \dot{x}_2^2 + C_3 \dot{x}_3^2 \\ &= C_1 (L_1 \dot{\theta})^2 + C_2 (L_2 \dot{\theta})^2 + C_3 (L_3 \dot{\theta})^2 \\ &= \underbrace{(C_1 L_1^2 + C_2 L_2^2 + C_3 L_3^2)}_{C_t} \dot{\theta}^2\end{aligned}$$

(2) Model: $\dot{E}_d = C_{eq} \dot{x}_1^2$

system:
$$\begin{aligned}\dot{E}_d &= C_1 \dot{x}_1^2 + C_2 \dot{x}_2^2 + C_3 \dot{x}_3^2 \\ &= (C_1 L_1^2 + C_2 L_2^2 + C_3 L_3^2) \dot{\theta}^2 \\ &= \underbrace{\frac{C_1 L_1^2 + C_2 L_2^2 + C_3 L_3^2}{L_1^2}}_{C_{eq}} \dot{x}_1^2\end{aligned}$$