

## PUMP SCALING LAWS

In this lesson, we will:

- Review **Dimensional Analysis** by applying it to **Pump Performance Curves**
- Discuss the **Pump Affinity Laws** and how to apply them to design pumps
- Do an example problem

### Dimensional Analysis of Pump Performance Parameters

Start with the functional relationship, and perform the method of repeating variables:

$gH = f(\dot{V}, D, \varepsilon, \omega, \rho, \mu)$ 
  
*gH occur together*
  
 $D = \text{pump dia}$ 
  
 $\omega \text{ in rad/s}$ 
  
 $[n \text{ in RPM}]$ 
  
 $\{\omega\} = \left\{\frac{1}{t}\right\}$ 
  
 $\{n\} = \left\{\frac{1}{t}\right\}$

Try it yourself – great review of dimensional analysis and the method of repeating variables:

$$\frac{gH}{\omega^2 D^2} = \text{function of} \left( \frac{\dot{V}}{\omega D^3}, \frac{\rho \omega D^2}{\mu}, \frac{\varepsilon}{D} \right)$$

A similar analysis with input brake horsepower (*bhp*) as a function of the same variables results in

$$\frac{bhp}{\rho \omega^3 D^5} = \text{function of} \left( \frac{\dot{V}}{\omega D^3}, \frac{\rho \omega D^2}{\mu}, \frac{\varepsilon}{D} \right)$$

Let's name these  $\Pi$ 's:

$$C_H = \frac{gH}{\omega^2 D^2} = \text{head coefficient}$$

$$\frac{\varepsilon}{D} = \text{roughness ratio}$$

$$C_P = \frac{bhp}{\rho \omega^3 D^5} = \text{power coefficient}$$

$$Re = \frac{\rho \omega D^2}{\mu} = \text{Reynolds number}$$

$$C_Q = \frac{\dot{V}}{\omega D^3} = \text{capacity coefficient}$$

*Q is used since Q = same as V*

So, we write

$$C_H = \text{function}(C_Q, Re, \varepsilon/D)$$

and

$$C_P = \text{function}(C_Q, Re, \varepsilon/D)$$

But for many pumps, effects of  $Re$  and roughness are small **at high  $Re$** , and thus,

$$C_H \approx \text{function}(C_Q) \quad \text{and} \quad C_P \approx \text{function}(C_Q)$$

Finally, **pump efficiency** is already dimensionless, and we write  $\eta_{\text{pump}}$  as

$$\eta_{\text{pump}} = \frac{\rho(\dot{V})(gH)}{bhp} = \frac{\rho(\omega D^3 C_Q)(\omega^2 D^2 C_H)}{\rho \omega^3 D^5 C_P} = \frac{C_Q C_H}{C_P} = \text{function}(C_Q)$$

$$\eta_{\text{pump}} = \text{fnc}(C_Q) \text{ also}$$

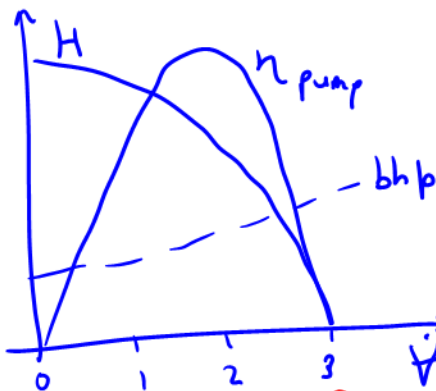
## Pump Affinity Laws

Definition of **affinity**: Inherent likeness or agreement.

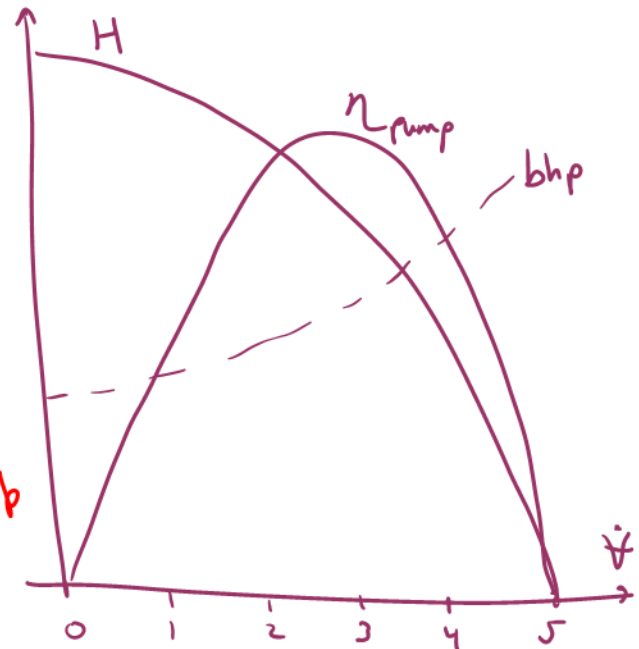
If two pumps are geometrically similar  $\therefore$  dynamically similar,  
then their pump performance curves fall on top of each other when plotted nondimensionally

E.g.,

PUMP A

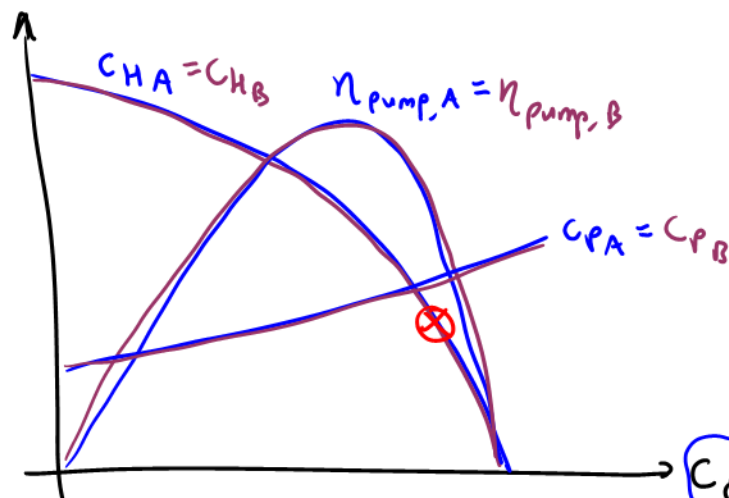


PUMP B



DIMENSIONAL PLOTS OF  $H$  &  $bhp$   
( $\eta_{pump}$  is nondimensional)

Re-draw these plots in nondimensional form



★ ONE SET OF  
PUMP  
PERFORMANCE  
CURVES  
APPLIES TO  
ALL SIMILAR  
PUMPS

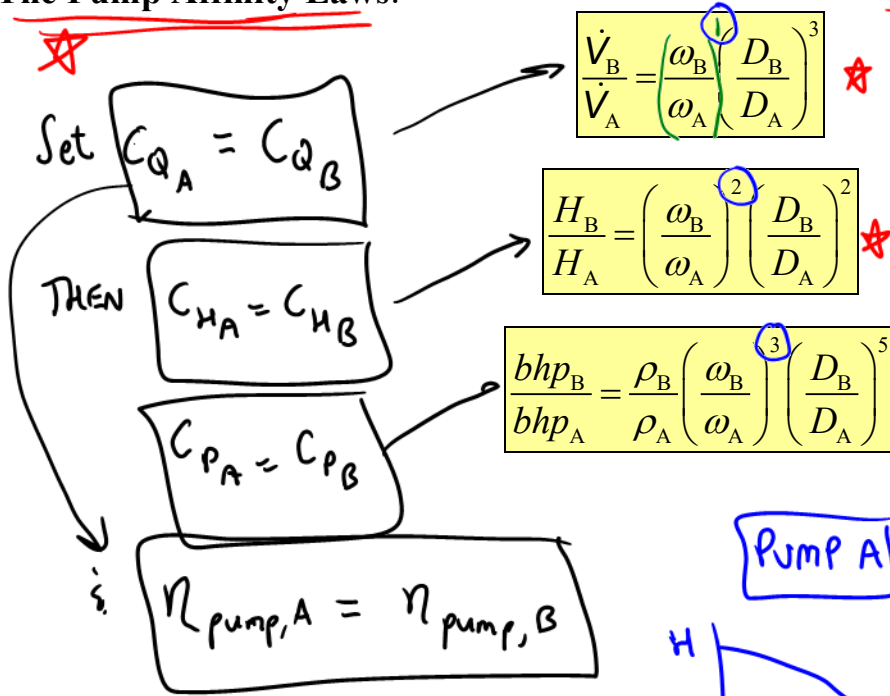
• Geometrically similar

• Dynamically similar

$C_Q$

+  $Re \propto \frac{E}{D}$  at small  $Re$

# The Pump Affinity Laws:

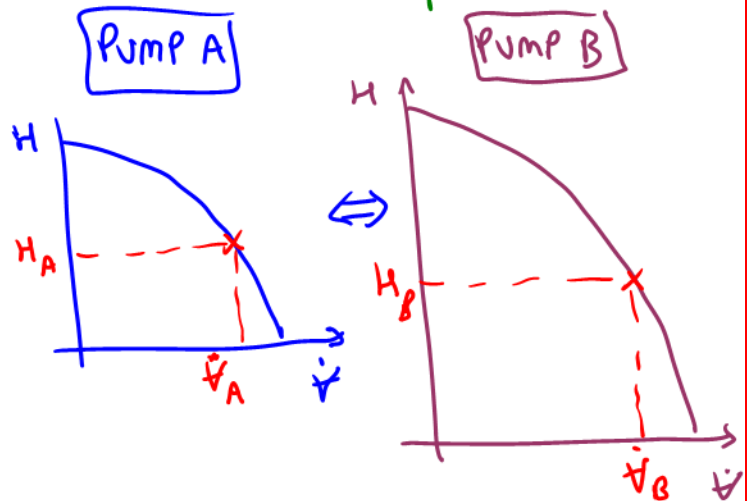


"VERY HARD PROBLEMS  
ARE AS EASY AS 1-2-3!"

## ★ PUMP AFFINITY LAWS

E.g., we double the RPM

- $\dot{V} \uparrow$  by factor of  $2^1 = 2$
- $H \uparrow$  ...  $2^2 = 4$
- $bhp \uparrow$  ...  $2^3 = 8$



At some operating point  
for Pump A, there is a  
corresponding operating point for Pump B

★ THESE ARE CALLED HOMOLOGOUS POINTS

If  $C_{Q_A} = C_{Q_B} \rightarrow C_{H_A} = C_{H_B} ; C_{P_A} = C_{P_B}$   
 $\eta_{\text{pump},A} = \eta_{\text{pump},B}$   
 @ Homologous points

$$C_{Q_A} = \frac{\dot{V}_A}{\omega_A D_A^3} = C_{Q_B} = \frac{\dot{V}_B}{\omega_B D_B^3} \Rightarrow \frac{\dot{V}_B}{\dot{V}_A} = \frac{\omega_B}{\omega_A} \left( \frac{D_B}{D_A} \right)^3$$

★ OUR FIRST AFFINITY LAW

### Example: Scaling up a pump using the affinity laws

**Given:** An existing pump (A): Fluid is water at 20°C,  $D_A = 6.50$  cm, and  $\dot{n}_A = 1500$  rpm. At BEP,  $\dot{V}_A = 455$  cm<sup>3</sup>/s at  $H_A = 1.44$  m. We are designing a new larger pump (B) that is geometrically similar with  $D_B = 8.20$  cm. It still uses water at 20°C, but rotates at a higher rpm,  $\dot{n}_B = 1750$  rpm.

**To do:** (a) Predict  $\dot{V}_B$  and  $H_B$  for operation of pump B at its BEP.  
(b) Estimate the % increase in required brake horsepower from pump A to pump B.

#### Solution:

(a) At homologous points, the two turbines are dynamically similar. Apply the affinity laws:

$$\frac{\dot{V}_B}{\dot{V}_A} = \frac{\omega_B}{\omega_A} \left( \frac{D_B}{D_A} \right)^3 \quad \frac{H_B}{H_A} = \left( \frac{\omega_B}{\omega_A} \right)^2 \left( \frac{D_B}{D_A} \right)^2 \quad \frac{bhp_B}{bhp_A} = \frac{\rho_B}{\rho_A} \left( \frac{\omega_B}{\omega_A} \right)^3 \left( \frac{D_B}{D_A} \right)^5$$

$$\dot{V}_B = \dot{V}_A \left( \frac{\omega_B}{\omega_A} \right) \left( \frac{D_B}{D_A} \right)^3 = \dot{V}_A \left( \frac{\dot{n}_B}{\dot{n}_A} \right) \left( \frac{D_B}{D_A} \right)^3$$

$\uparrow \dot{n}?$   $\left( \frac{2\pi \text{ rad}}{\text{rot}} \right) \left( \frac{1 \text{ min}}{60 \text{ s}} \right)$

$$\dot{V}_B = \left( 455 \frac{\text{cm}^3}{\text{s}} \right) \left( \frac{1750 \text{ rpm}}{1500 \text{ rpm}} \right) \left( \frac{8.20 \text{ cm}}{6.50 \text{ cm}} \right)^3 = 1065.76 \frac{\text{cm}^3}{\text{s}}$$

$$\dot{V}_B = 1070 \frac{\text{cm}^3}{\text{s}}$$

$$H_B = H_A \left( \frac{\dot{n}_B}{\dot{n}_A} \right)^2 \left( \frac{D_B}{D_A} \right)^2 \rightsquigarrow H_B = 3.12 \text{ m}$$

$$(b) \frac{bhp_B}{bhp_A} = \frac{\rho_B}{\rho_A} \left( \frac{\dot{n}_B}{\dot{n}_A} \right)^3 \left( \frac{D_B}{D_A} \right)^5 = (1) \left( \frac{1750}{1500} \right)^3 \left( \frac{8.20}{6.50} \right)^5 = 5.0739$$

$\uparrow$  Here  $\rho_B = \rho_A$

$\approx 5$  times as much power !!

$$\% \text{ increase} = \frac{bhp_B - bhp_A}{bhp_A} \times 100\% = \left( \frac{bhp_B}{bhp_A} - 1 \right) \times 100\% = 407\% \text{ increase in bhp}$$