

Today, we will:

- Do another example – the stream function.
- Discuss the differential equation for momentum in fluid flow: **The Navier-Stokes eq.**
- Do some example problems – Navier-Stokes equation

Derivation of the Navier-Stokes Equation (Section 9-5, Çengel and Cimbala)

We begin with the general differential equation for conservation of linear momentum, i.e., *Cauchy's equation*, which is valid for any kind of fluid,

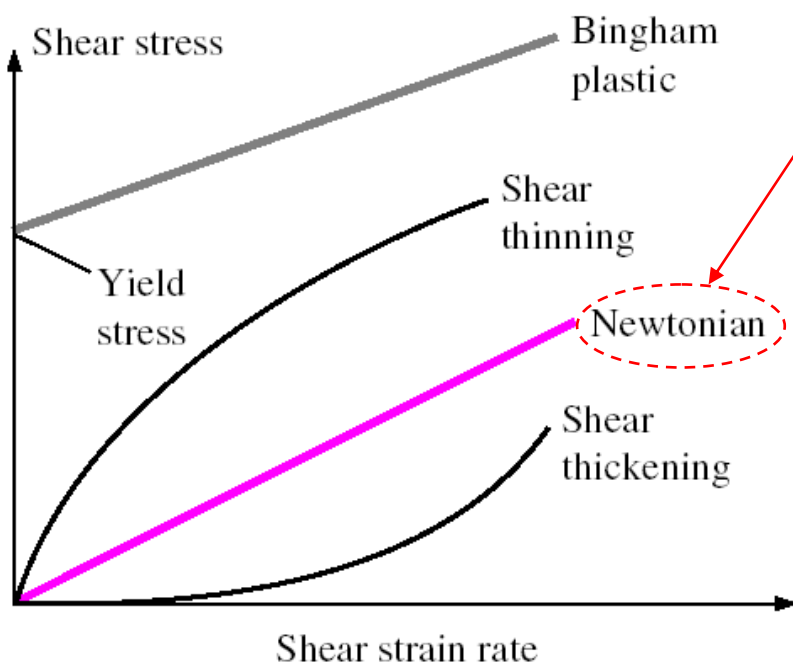
Cauchy's equation:

$$\rho \frac{D\vec{V}}{Dt} = \rho \vec{g} + \vec{\nabla} \cdot \sigma_{ij} \quad (9-50)$$

Stress tensor

The problem is that the stress tensor σ_{ij} needs to be written in terms of the primary unknowns in the problem in order for Cauchy's equation to be useful to us. The equations that relate σ_{ij} to other variables in the problem – velocity, pressure, and fluid properties – are called *constitutive equations*. There are different constitutive equations for different kinds of fluids.

Types of fluids:



For *Newtonian fluids*, the shear stress is linearly proportional to the shear strain rate.

Examples of Newtonian fluids: water, air, oil, gasoline, most other common fluids.

FIGURE 9-37

Rheological behavior of fluids—shear stress as a function of shear strain rate.

Some examples of non-Newtonian fluids:

- Paint (*shear thinning* or *pseudo-plastic*)
- Toothpaste (*Bingham plastic*)
- Quicksand (*shear thickening* or *dilatant*).

We consider only Newtonian fluids in this course.

For *Newtonian fluids* (see text for derivation), it turns out that

$$\sigma_{ij} = \begin{pmatrix} -P & 0 & 0 \\ 0 & -P & 0 \\ 0 & 0 & -P \end{pmatrix} + \begin{pmatrix} 2\mu \frac{\partial u}{\partial x} & \mu \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) & \mu \left(\frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right) \\ \mu \left(\frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right) & 2\mu \frac{\partial v}{\partial y} & \mu \left(\frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} \right) \\ \mu \left(\frac{\partial w}{\partial x} + \frac{\partial u}{\partial z} \right) & \mu \left(\frac{\partial w}{\partial y} + \frac{\partial v}{\partial z} \right) & 2\mu \frac{\partial w}{\partial z} \end{pmatrix} \quad (9-57)$$

We have achieved our goal of writing σ_{ij} in terms of pressure P , velocity components u, v , and w , and fluid viscosity μ .

Now we plug this expression for the stress tensor σ_{ij} into Cauchy's equation. The result is the famous *Navier-Stokes equation*, shown here for incompressible flow.

Incompressible Navier-Stokes equation:

$$\text{Navier-Stokes equation: } \rho \frac{D\vec{V}}{Dt} = -\vec{\nabla}P + \rho\vec{g} + \mu \nabla^2 \vec{V} \quad (9-60)$$

To solve fluid flow problems, we need both the continuity equation and the Navier-Stokes equation. Since it is a vector equation, the Navier-Stokes equation is usually split into three components in order to solve fluid flow problems. In Cartesian coordinates,

Incompressible continuity equation:

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0 \quad (9-61a)$$

x-component of the incompressible Navier-Stokes equation:

$$\rho \left(\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right) = -\frac{\partial P}{\partial x} + \rho g_x + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right) \quad (9-61b)$$

y-component of the incompressible Navier-Stokes equation:

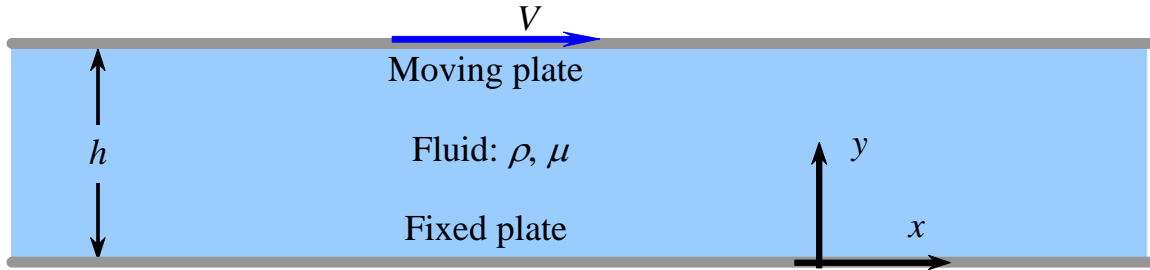
$$\rho \left(\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right) = -\frac{\partial P}{\partial y} + \rho g_y + \mu \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right) \quad (9-61c)$$

z-component of the incompressible Navier-Stokes equation:

$$\rho \left(\frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} \right) = -\frac{\partial P}{\partial z} + \rho g_z + \mu \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right) \quad (9-61d)$$

Example Problem – Exact Solution for Couette Flow

Given: Steady, incompressible, laminar flow in the x - y plane between two infinite parallel plates.



Assumptions and approximations:

1. The flow is steady [$\partial/\partial t$ of anything = 0].
2. The flow is two-dimensional in the x - y plane [$\partial/\partial z$ of anything = 0, $w = 0$].
3. Gravity effects are negligible or ignored.
4. The flow is fully developed [$\partial/\partial x$ of any velocity = 0 – velocity does not change with x].
5. Pressure is constant everywhere.

To do: Calculate the velocity field.

Solution: [to be done in class]