

Today, we will:

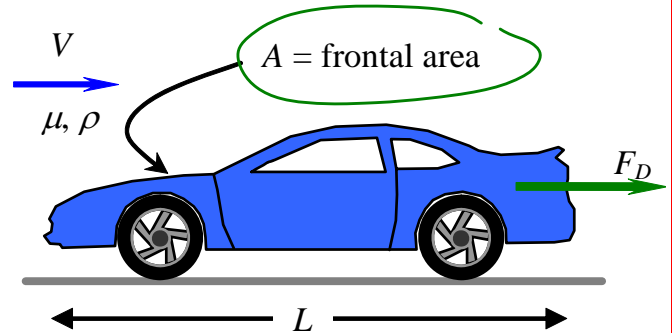
- Finish the example problem from last lecture.
- Do some more example problems – dimensional analysis
- Discuss experimental testing and incomplete similarity

Example: Dimensional analysis – Car drag

Given: The drag force F_D on a car is a function of four variables: air velocity V , air density ρ , air viscosity μ , and the length L of the car.

To do: Express this relationship in terms of nondimensional parameters.

Solution: We follow the six steps for the method of repeating variables.



See previous lecture. We were in the middle of step 5, and had

$$\Pi_1 = \text{dependent Pi} = F_D V^a L^b \rho^c = F_D V^{-2} L^{-2} \rho^{-1} = \frac{F_D}{\rho V^2 L^2}$$

Modify or manipulate this Π if necessary to match one of the named established Π 's (see Table 7-5)

Modified $\Pi_1 = C_D = \text{drag coefficient} = \frac{F_D}{\frac{1}{2} \rho V^2 A}$

$$\Pi_2 = \mu V^a L^b \rho^c$$

$$\{\Pi_2\} = \{1\} = \{m^0 L^0 t^0\} = \left\{ \mu V^a L^b \rho^c \right\}$$

$$\left\{ \left(\frac{m}{L t} \right) \left(\frac{L}{t} \right)^a (L)^b \left(\frac{m}{L^3} \right)^c \right\}$$

$$\{m^0 L^0 t^0\} = \{m^1 L^{-1} t^{-1} L^a t^{-a} L^b m^c L^{-3c}\}$$

equating coefficients: $m: m^0 = m^1 m^c \rightarrow 0 = 1 + c \rightarrow \boxed{c = -1}$

$t: t^0 = t^{-1} t^{-a} \rightarrow 0 = -1 - a \rightarrow \boxed{a = -1}$

$L: L^0 = L^{-1} L^a L^b L^{-3c} \rightarrow 0 = -1 + a + b - 3c \rightarrow \boxed{b = -1}$

$$\Pi_2 = \frac{\mu}{\rho V L}$$

Guidelines for Manipulating the Π Parameters

There are several guidelines for manipulating the Π parameters. These guidelines are listed concisely in Table 7-4 in the text, as summarized below: See Table 7-4 for more details.

1. We may impose a constant (dimensionless) exponent on a Π or perform a functional operation on a Π .
2. We may multiply a Π by a pure (dimensionless) constant.
3. We may form a product (or quotient) of any Π with any other Π in the problem to replace one of the Π 's.
4. We may use any of guidelines 1 to 3 in combination.
5. We may substitute a dimensional parameter in the Π with other parameter(s) of the same dimensions.

↓ Here we sub. A for L^2
 $\{L^2\}$ $\{L^2\}$

The goal is to get each Π into a form that looks like one of the common *established* nondimensional parameters that are listed in Table 7-5 in the text. Some of the most popular and often-used ones are listed below. A more exhaustive list is given in the text.

TABLE 7-5

Some common established nondimensional parameters or Π 's encountered in fluid mechanics and heat transfer*

Name	Definition	Ratio of Significance
Darcy friction factor	$f = \frac{8\tau_w}{\rho V^2}$ nondim constant	$\frac{\text{Wall friction force}}{\text{Inertial force}}$
Drag coefficient	$C_D = \frac{F_D}{\frac{1}{2}\rho V^2 A}$ or $\frac{F_0}{\rho V^2 L^2}$	$\frac{\text{Drag force}}{\text{Dynamic force}}$
Froude number	$Fr = \frac{V}{\sqrt{gL}}$ (sometimes $\frac{V^2}{gL}$)	$\frac{\text{Inertial force}}{\text{Gravitational force}}$
Lift coefficient	$C_L = \frac{F_L}{\frac{1}{2}\rho V^2 A}$	$\frac{\text{Lift force}}{\text{Dynamic force}}$
Mach number	Ma (sometimes M) = $\frac{V}{c}$	$\frac{\text{Flow speed}}{\text{Speed of sound}}$
Reynolds number	$Re = \frac{\rho VL}{\mu} = \frac{VL}{\nu}$	$\frac{\text{Inertial force}}{\text{Viscous force}}$
Strouhal number	St (sometimes S or Sr) = $\frac{fL}{V}$	$\frac{\text{Characteristic flow time}}{\text{Period of oscillation}}$

★ Reynolds number is the most important nondimensional parameter in fluid mechanics.

Manipulate $\rightarrow \boxed{\pi_2 = Re = \frac{\rho V L}{\mu}} = \underline{\underline{\text{Reynolds \#}}}$

I took $(\pi_2)^{-1}$ to get Re

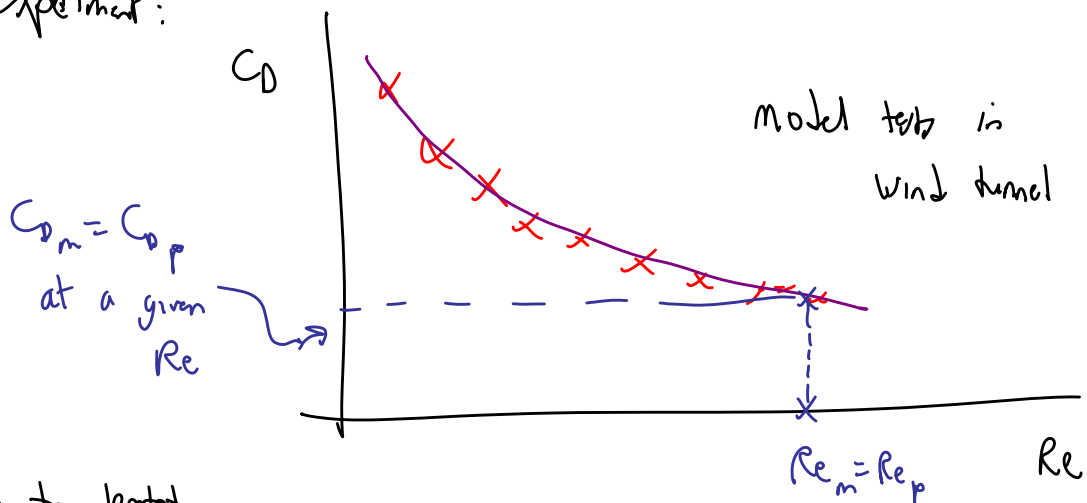
Step 6 \rightarrow Write the functional relationship: $\pi_1 = \text{func}(\pi_2, \pi_3, \dots)$

here $\pi_1 = \text{func}(\pi_2)$ or $\boxed{C_D = \text{func}(Re)}$ *

Originally $\boxed{F_D = \text{func}(V, L, \rho, \mu)}$ 5 parameters, 4 indep. variables

Now we have $\star \boxed{C_D = \text{func}(Re)}$ 2 parameters, 1 indep. variable

experiment:



Scale up to prototype

Given: Prototype car

$V_p = 50 \text{ mph}$
 $L_p = 15 \text{ ft}$

, Predict $F_{D,p}$
air has ρ, μ

Model car

$\frac{1}{5}$ scale $\rightarrow L_m = 3 \text{ ft}$

same air properties

To do: (a) How fast to run the wind tunnel to achieve dynamic similarity? (V_m)

Soln: Needs to have $\pi_{2m} = \pi_{2p} \rightarrow Re_m = Re_p$

$$Re_m = \frac{\rho_m L_m V_m}{\mu_m} = Re_p = \frac{\rho_p L_p V_p}{\mu_p}$$

Solve $V_m = \frac{V_p \frac{\rho_p}{\rho_m} \frac{L_p}{L_m} \frac{\mu_m}{\mu_p}}{1} = 250 \text{ mph} = V_m$

\downarrow 50 mph \downarrow 5
 1 1

Since $\pi_{2m} = \pi_{2p}$, then $\pi_{1m} = \pi_{1p}$ (dynamic similarity)

Here, since $Re_m = Re_p$, then $C_{Dm} = C_{Dp}$

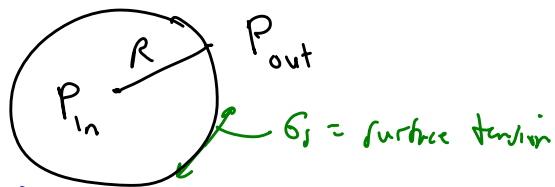
$$\frac{F_{Dp}}{\frac{1}{2} \rho_p V_p^2 A_p} = \frac{F_{Dm}}{\frac{1}{2} \rho_m V_m^2 A_m}$$

\circlearrowleft We measure this \rightarrow Suppose it is 25.0 lbf

Solve $F_{Dp} = F_{Dm} \left(\frac{\rho_p}{\rho_m} \right) \left(\frac{V_p}{V_m} \right)^2 \left(\frac{A_p}{A_m} \right)$

$$(25.0 \text{ lbf})(1) \left(\frac{50 \text{ mph}}{250 \text{ mph}} \right)^2 (5^2) = 25.0 \text{ lbf} = F_{Dp}$$

Example Soap Bubble



Given:

$$\Delta P = P_{in} - P_{out} = f(\sigma_s, R)$$

To do:

Find the functional relationship between ΔP ; σ_s ; R

Step 1 $\Delta P =$ func (σ_s, R) $n=3$

Step 2 $\left\{ \frac{m}{t^2 L} \right\}$ $\left\{ \frac{m}{t^2} \right\}$ $\{L\}$

Step 3 find the reduction j Try $j=3$ since I have m, L, t

\therefore We expect $k = n - j = 3 - 3 = 0$ π 's ??? ~~X~~ impossible!

Let's try instead $j = j - 1 \rightarrow \underline{j=2}$, $k = n - j = 3 - 2 = \underline{1}$ π

Step 4 select j repeating variables \rightarrow pick σ_s, R

Step 5 solve for the π 's

$$\pi_1 = \Delta P \sigma_s^a R^b$$

$$\{m^0 L^0 t^0\} = \left\{ \frac{m}{t^2 L} \left(\frac{m}{t^2} \right)^a (L)^b \right\}$$

$$m: 0 = 1 + a \rightarrow \underline{a = -1}$$

$$L: 0 = -1 + b \rightarrow \underline{b = 1}$$

$$t: 0 = -2 - 2a \rightarrow \underline{a = -1}$$

agree

$$\pi_1 = \Delta P \sigma_s^{-1} R^1$$

$$\pi_1 = \frac{\Delta P R}{\sigma_s}$$

\rightarrow here, no "manipulation" necessary

Step 6 final relationship: $\pi_1 = \text{func}(\text{nothing})!$ $\pi_1 = \text{constant}$

$$\Delta P = \frac{\text{const} \cdot \sigma_s}{R}$$

recall, the constant = 4

$$\Delta P = \frac{4 \sigma_s}{R}$$

[Dim. anal. is not able to give us the value of the constant. Need on experiment or analysis]

Example: Dimensional analysis – shaft power

Given: The output power \dot{W} of a spinning shaft is a function of torque T and angular velocity ω . *(NOTE: I DID THIS PROBLEM AFTER CLASS)*

To do: Express the relationship between \dot{W} , T , and ω in dimensionless form.

Solution:

Step 1: List: $\dot{W} = fnc(T, \omega)$ $n = 3$

Step 2: Dim's $\left\{ \frac{ML^2}{t^3} \right\}$ $\left\{ \frac{ML^2}{t^2} \right\}$ $\left\{ \frac{1}{t} \right\}$

Note: Rank is dimensionless
 $\left\{ \omega \right\} = \left\{ \frac{rad}{s} \right\} = \left\{ \frac{1}{t} \right\}$

Step 3: pick $\bar{j} = 3$, expect $k = n - \bar{j} = 3 - 3 = 0$ π 's ~~X~~ impossible!
pick $\bar{j} = 2$ instead. expect $k = n - \bar{j} = 3 - 2 = \underline{\underline{1}}$ π

pick 2 repeating variables \rightarrow pick T & ω

Step 4: calculate π 's:

$$\pi_1 = \dot{W} T^a \omega^b$$

Step 5: $\left\{ \pi_1 \right\} = \left\{ m^0 L^0 t^0 \right\} = \left\{ \left(\frac{ML^2}{t^3} \right) \left(\frac{ML^2}{t^2} \right)^a \left(\frac{1}{t} \right)^b \right\}$

Equate exponents: m: $1 + a = 0 \rightarrow \underline{a = -1}$ } agree \odot

L: $2 + 2a = 0 \rightarrow a = -1$

t: $-3 - 2a - b = 0 \rightarrow \underline{b = -1}$

So, $\pi_1 = \frac{\dot{W}}{T\omega}$

Step 6: functional relationship $\rightarrow \pi_1 = fnc(\pi_2, \pi_3, \dots, \pi_k)$

Here $\pi_1 = fnc(\text{nothing}) = \text{constant}$.

$\therefore \dot{W} = \text{const} \omega T$

Our answer is correct to within an unknown constant, without knowing any physics!

(The constant turns out to = 1, but we can't know that from Dim. Anal.)