

Static Force Analysis for Skid Loader - Scalar

A trunnion mount hydraulic cylinder actuates the arm of a skid steer loader as shown below. At this position, $e = 40$ inches, $\theta = 61.131^\circ$, $\dot{e} = -12$ ips, $\dot{\theta} = -0.3625$ rad/s.

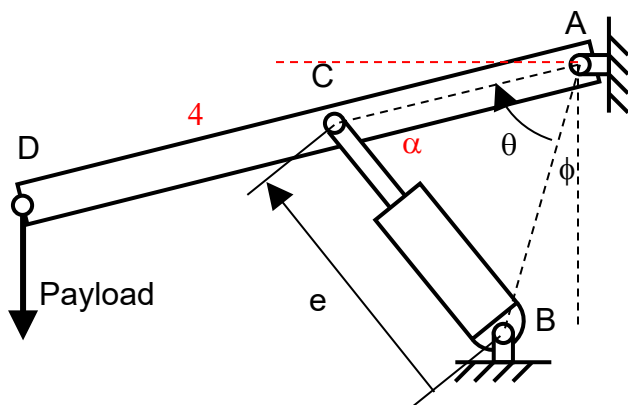
Determine the force on the hydraulic cylinder required to lower an 800 lbf payload attached to point D by a cable. The payload moves with constant velocity at the position shown. You may neglect the effects of friction. The weight of the arm and cylinder are small compared to the payload. Show your work.

$F_{\text{CYLINDER}} = 2261.9 \text{ lbf up/left}$

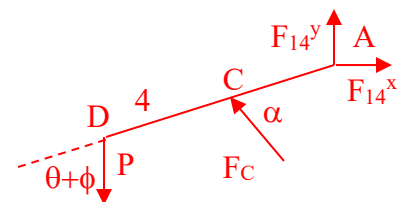
$$\sin \alpha / AB = \sin \theta / e$$

$$\alpha = 52.01^\circ$$

$$\theta + \phi = 77.131^\circ$$



Not to scale
 $AB = 36$ inches
 $AC = 42$ inches
 $AD = 96$ inches
 $\phi = 16^\circ$



$$\sum M \text{ on 4 about A CCW} +$$

$$-(F_c \sin \alpha) AC + (P \sin(\theta + \phi)) AD = 0$$

$$F_c = 2261.9 \text{ lbf}$$

What corresponding hydraulic pressure would be required for a cylinder with a 3 inch DIA bore?

$P_{\text{CYLINDER}} = 320 \text{ psi}$

$$A = \pi D^2 / 4 = 7.069 \text{ in}^2$$

Is this value reasonable? Why?

OK, industrial hydraulics often go to 3000 psi

If you include friction between the piston and cylinder wall, will it increase or decrease your computation for pressure.

increase decrease Why?

pressure pushes up
 friction force will be up opposing piston motion

What value would you use for the coefficient of friction between the piston and cylinder wall?

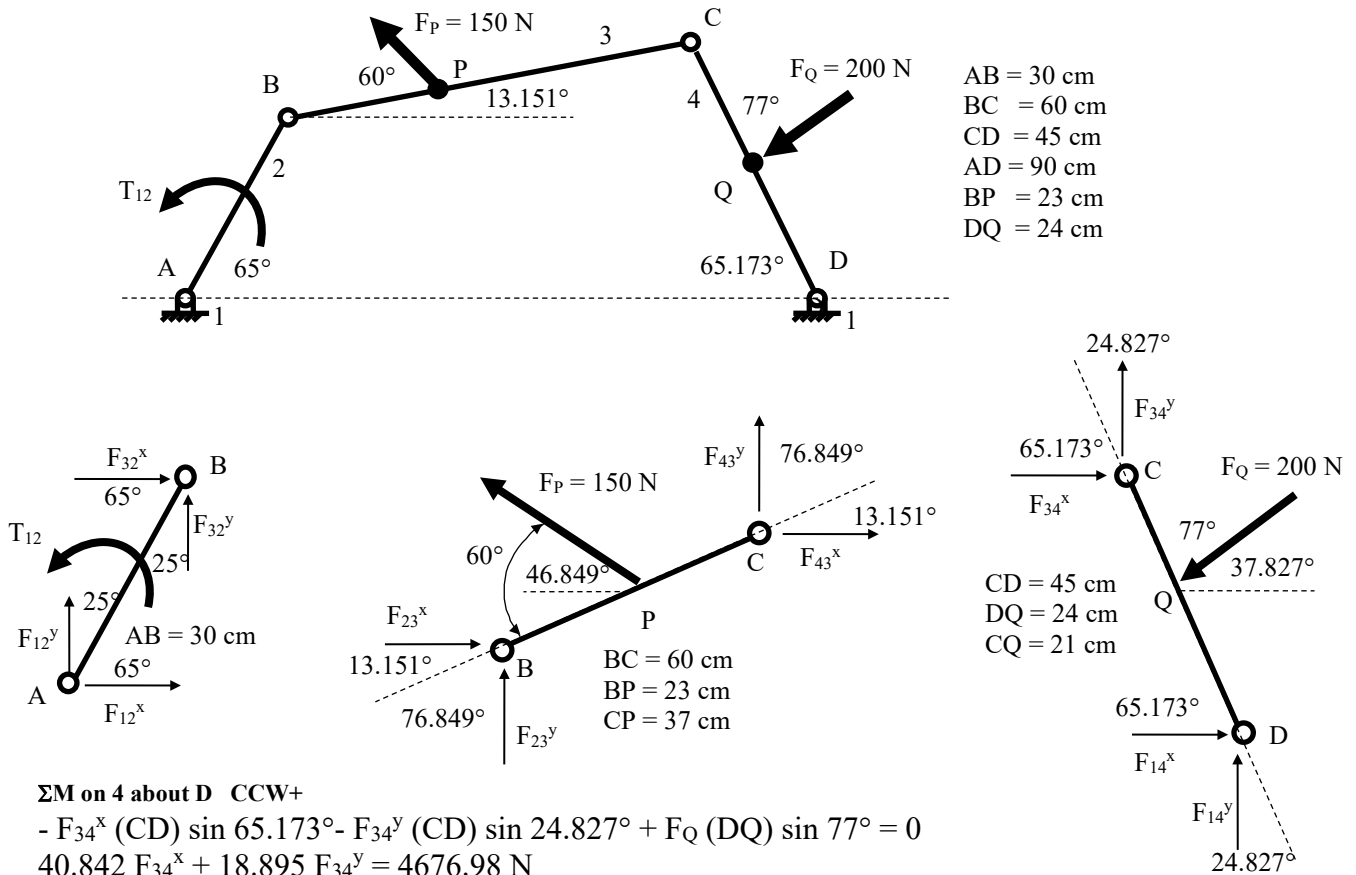
$\mu = 0.1$ lubricated Why?

Should your analysis be different if the cylinder were retracting at constant velocity instead of the payload moving at constant velocity?

yes no Why?

constant $\dot{e}$ means $\dot{\theta}$ will not be constant means velocity of the payload will not be constant, therefore must account for acceleration of payload mass

Static Force Analysis for Four Bar - Scalar



ΣM on 4 about D CCW+

$$-F_{34}^x (CD) \sin 65.173^\circ - F_{34}^y (CD) \sin 24.827^\circ + F_Q (DQ) \sin 77^\circ = 0$$

$$40.842 F_{34}^x + 18.895 F_{34}^y = 4676.98 \text{ N}$$

ΣM on 3 about B CCW+

$$-F_{43}^x (BC) \sin 13.151^\circ + F_{43}^y (BC) \sin 76.849^\circ + F_P (BP) \sin 60^\circ = 0$$

$$-13.651 F_{43}^x + 58.426 F_{43}^y = -2987.78 \text{ N}$$

$$\begin{bmatrix} 40.842 & 18.895 \\ +13.651 & -58.426 \end{bmatrix} \begin{Bmatrix} F_{34}^x \\ F_{34}^y \end{Bmatrix} = \begin{Bmatrix} 4676.98 \text{ N} \\ -2987.78 \text{ N} \end{Bmatrix} \quad \begin{Bmatrix} F_{34}^x \\ F_{34}^y \end{Bmatrix} = \begin{Bmatrix} 81.993 \text{ N} \\ 70.295 \text{ N} \end{Bmatrix}$$

ΣF on 4 right +

$$F_{34}^x - F_Q \cos 37.827^\circ + F_{14}^x = 0 \quad F_{14}^x = 75.980 \text{ N}$$

ΣF on 4 up +

$$F_{34}^y - F_Q \sin 37.827^\circ + F_{14}^y = 0 \quad F_{14}^y = 52.360 \text{ N}$$

ΣF on 3 right +

$$F_{43}^x - F_P \cos 46.849^\circ + F_{23}^x = 0 \quad F_{23}^x = 184.580 \text{ N}$$

ΣF on 3 up +

$$F_{43}^y + F_P \sin 46.849^\circ + F_{23}^y = 0 \quad F_{23}^y = -39.138 \text{ N}$$

ΣM on 2 about A CCW+

$$-F_{32}^x (AB) \sin 65^\circ + F_{32}^y (AB) \sin 25^\circ + T_{12} = 0$$

$$T_{12} = -5514.8 \text{ N.cm}$$

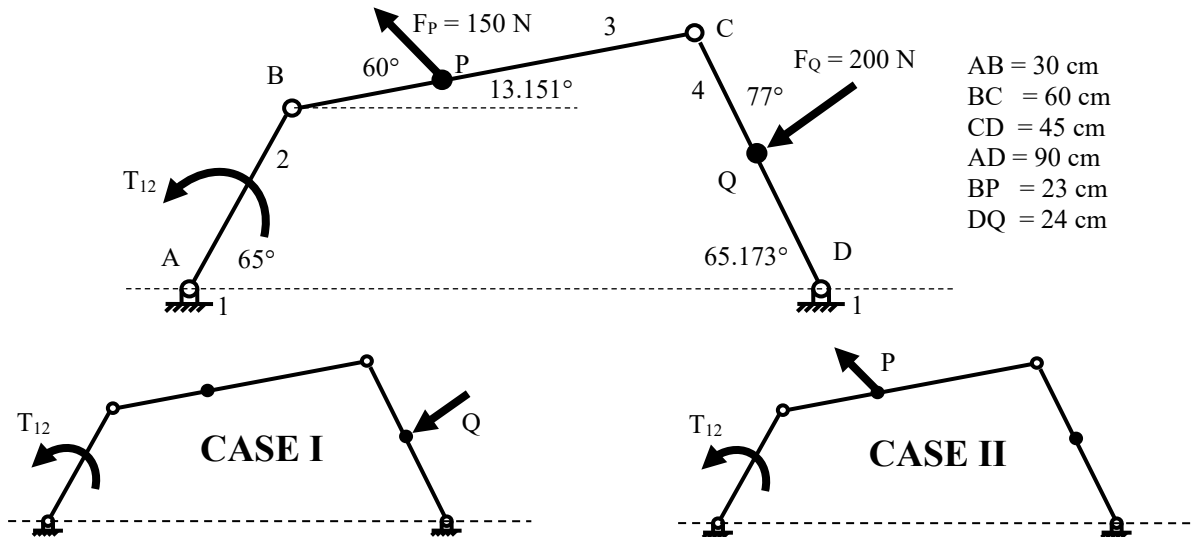
ΣF on 2 right +

$$F_{12}^x + F_{32}^x = 0 \quad F_{12}^x = 184.580 \text{ N}$$

ΣF on 2 up

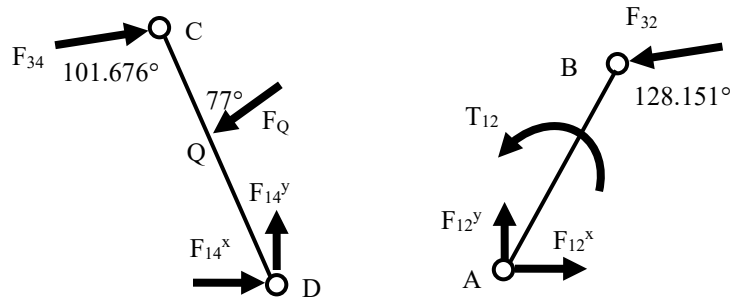
$$F_{12}^y + F_{32}^y = 0 \quad F_{12}^y = -39.138 \text{ N}$$

Static Force Analysis for Four Bar - Superposition



CASE I - F_{34} parallel to BC

$$\begin{aligned} \Sigma M \text{ on 4 about D CCW} + \\ - F_{34} \sin 101.676^\circ CD + F_Q \sin 77^\circ DQ &= 0 \\ F_{34} &= 106.129 \text{ N} = F_{32} \\ \Sigma M \text{ on 2 about A CCW} + \\ + F_{32} \sin 128.151^\circ AB + T_{12} &= 0 \\ T_{12} &= -2503.75 \text{ N.cm} \end{aligned}$$



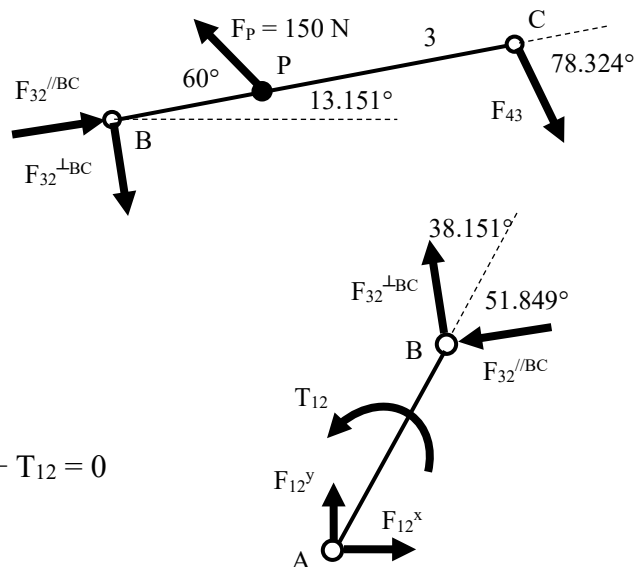
CASE II - F_{43} parallel to CD

$$\begin{aligned} \Sigma M \text{ on 3 about B CCW} + \\ - F_{43} \sin 101.676^\circ BC + F_P \sin 60^\circ BP &= 0 \\ F_{43} &= 50.849 \text{ N} \\ \Sigma F \text{ on 3 //BC right} + \\ + F_{43} \cos 78.324^\circ - F_P \cos 60^\circ + F_{32}^{\text{//BC}} &= 0 \\ F_{32}^{\text{//BC}} &= 64.709 \text{ N} \end{aligned}$$

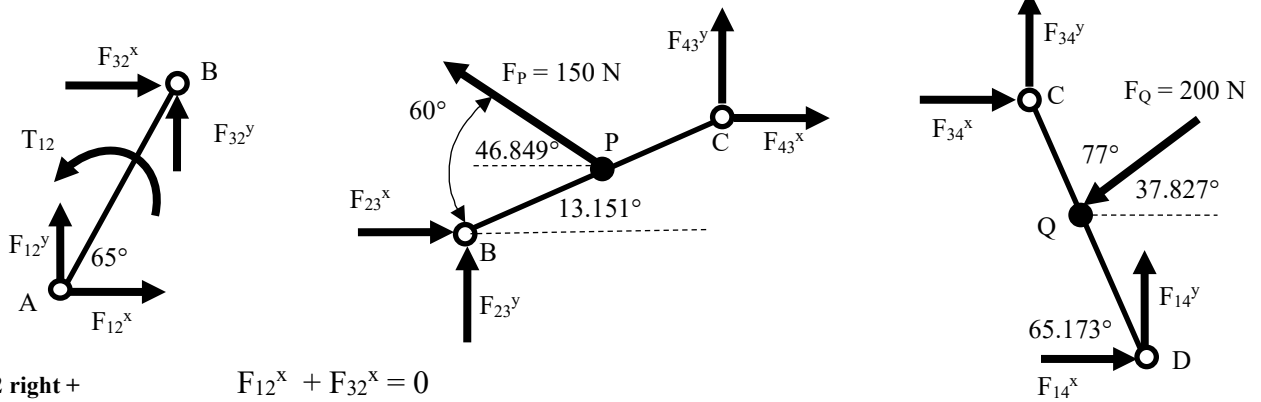
$$\begin{aligned} \Sigma F \text{ on 3 } \perp BC \text{ up} + \\ - F_{43} \sin 78.324^\circ + F_P \cos 60^\circ - F_{32}^{\perp BC} &= 0 \\ F_{32}^{\perp BC} &= 80.107 \text{ N} \end{aligned}$$

$$\begin{aligned} \Sigma M \text{ on 2 about A CCW} + \\ + F_{32}^{\text{//BC}} \sin 51.849^\circ AB + F_{32}^{\perp BC} \sin 38.151^\circ AB + T_{12} &= 0 \\ T_{12} &= -3011.14 \text{ N.cm} \end{aligned}$$

$$T_{12} = T_{12}(\text{Case I}) + T_{12}(\text{Case II}) = -5514.89 \text{ N.cm}$$



Static Force Analysis for Four Bar - Matrix



$$\begin{aligned}
 \Sigma F \text{ on 2 right} + & F_{12}^x + F_{32}^x = 0 \\
 \Sigma F \text{ on 2 up} + & F_{12}^y + F_{32}^y = 0 \\
 \Sigma M \text{ on 2 about A CCW} + & -F_{32}^x r_{B/A}^y + F_{32}^y r_{B/A}^x + T_{12} = 0 \\
 \\
 \Sigma F \text{ on 3 right} + & F_{23}^x + F_{43}^x + F_P^x = 0 \\
 \Sigma F \text{ on 3 up} + & F_{23}^y + F_{43}^y + F_P^y = 0 \\
 \Sigma M \text{ on 3 about P CCW} + & F_{23}^x r_{B/P}^y - F_{23}^y r_{B/P}^x - F_{43}^x r_{C/P}^y + F_{43}^y r_{C/P}^x = 0 \\
 \\
 \Sigma F \text{ on 4 right} + & F_{34}^x + F_{14}^x + F_Q^x = 0 \\
 \Sigma F \text{ on 4 up} + & F_{34}^y + F_{14}^y + F_Q^y = 0 \\
 \Sigma M \text{ on 4 about Q CCW} + & -F_{34}^x r_{C/Q}^y - F_{34}^y r_{C/Q}^x + F_{14}^x r_{D/Q}^y + F_{14}^y r_{D/Q}^x = 0
 \end{aligned}$$

$$\begin{bmatrix}
 1 & 0 & -1 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 1 & 0 & -1 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & r_{B/A}^y & -r_{B/A}^x & 0 & 0 & 0 & 0 & 1 \\
 0 & 0 & 1 & 0 & -1 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 1 & 0 & -1 & 0 & 0 & 0 \\
 0 & 0 & r_{B/P}^y & -r_{B/P}^x & r_{C/P}^y & -r_{C/P}^x & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & 1 & 0 & 1 & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & 1 & 0 & 1 & 0 \\
 0 & 0 & 0 & 0 & -r_{C/Q}^y & -r_{C/Q}^x & r_{D/Q}^y & r_{D/Q}^x & 0
 \end{bmatrix}
 \begin{bmatrix}
 F_{12}^x \\
 F_{12}^y \\
 F_{23}^x \\
 F_{23}^y \\
 F_{34}^x \\
 F_{34}^y \\
 F_{14}^x \\
 F_{14}^y \\
 T_{12}
 \end{bmatrix}
 =
 \begin{bmatrix}
 0 \\
 0 \\
 0 \\
 -F_P^x \\
 -F_P^y \\
 0 \\
 -F_Q^x \\
 -F_Q^y \\
 0
 \end{bmatrix}$$

$$\bar{F}_P = 150 \text{ N @ } 133.151^\circ = -102.589 + j 109.433 \text{ N}$$

$$\bar{F}_Q = 200 \text{ N @ } 217.827^\circ = -157.973 - j 122.656 \text{ N}$$

$$B/A = 30 \text{ cm @ } 65^\circ = 12.678 + j 27.189 \text{ cm}$$

$$B/P = 23 \text{ cm @ } 193.151^\circ = -22.396 - j 5.233 \text{ cm}$$

$$C/P = 37 \text{ cm @ } 13.151^\circ = 36.030 + j 8.418 \text{ cm}$$

$$C/Q = 21 \text{ cm @ } 114.827^\circ = -8.817 + j 19.059 \text{ cm}$$

$$D/Q = 24 \text{ cm @ } -65.173^\circ = 10.077 - j 21.782 \text{ cm}$$

$$\begin{bmatrix}
 1 & 0 & -1 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 1 & 0 & -1 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 27.189 & -12.678 & 0 & 0 & 0 & 0 & 1 \\
 0 & 0 & 1 & 0 & -1 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 1 & 0 & -1 & 0 & 0 & 0 \\
 0 & 0 & -5.233 & 22.396 & 8.418 & -36.030 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & 1 & 0 & 1 & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & 1 & 0 & 1 & 0 \\
 0 & 0 & 0 & 0 & -19.059 & 8.817 & -21.782 & 10.077 & 0
 \end{bmatrix}
 \begin{Bmatrix}
 F_{12}^x \\
 F_{12}^y \\
 F_{23}^x \\
 F_{23}^y \\
 F_{34}^x \\
 F_{34}^y \\
 F_{14}^x \\
 F_{14}^y \\
 T_{12}
 \end{Bmatrix}
 =
 \begin{Bmatrix}
 0 \\
 0 \\
 0 \\
 102.589 \\
 -109.433 \\
 0 \\
 157.973 \\
 122.656 \\
 0
 \end{Bmatrix}$$

$$\begin{Bmatrix}
 F_{12}^x \\
 F_{12}^y \\
 F_{23}^x \\
 F_{23}^y \\
 F_{34}^x \\
 F_{34}^y \\
 F_{14}^x \\
 F_{14}^y \\
 T_{12}
 \end{Bmatrix}
 =
 \begin{Bmatrix}
 184.59 \text{ N} \\
 -39.14 \text{ N} \\
 184.59 \text{ N} \\
 -39.14 \text{ N} \\
 82.00 \text{ N} \\
 70.29 \text{ N} \\
 75.98 \text{ N} \\
 52.36 \text{ N} \\
 -5514.89 \text{ N.cm}
 \end{Bmatrix}$$

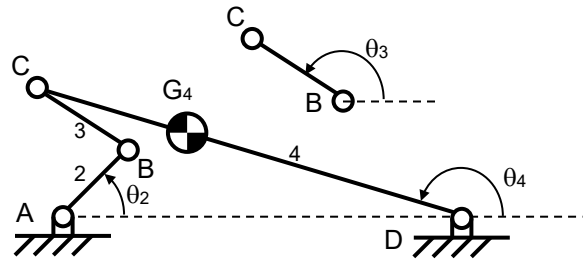
Static Force Analysis for Pushups - Matrix

A person doing pushups is represented with wrists A, elbows B, shoulders C, and toes D. Mass of the torso/legs is 180 lbm.

Use the additional assumptions:

- All muscular effort is provided by triceps as torque T_{32} across the elbows.
- Angular velocity of link 4 is constant at this position
- Weight of the arms is negligible compared to weight of the torso/legs.
- Friction is negligible at A, B, C and D.
- No muscular torque is generated at A, C and D.

AD = 52 inch	$\theta_2 = 45^\circ$
AB = 12 in	$\theta_3 = 149.14^\circ$
BC = 14 in	$\theta_4 = 164.24^\circ$
CD = 57.7 in	$\omega_2 = 0.5 \text{ rad/sec}$
DG ₄ = 39 in	$\omega_3 = -1.435 \text{ rad/sec}$
	$\omega_4 = -0.387 \text{ rad/sec}$



Determine angular velocity across the elbows $\omega_{2/3}$ for the position and velocity provided above.

$$\omega_{2/3} = \omega_2 - \omega_3 = +1.935 \text{ rad/sec}$$

Determine elbow torque T_{32} for the position and velocity provided above.

$$T_{32} = 1351.1 \text{ in.lbf CCW}$$

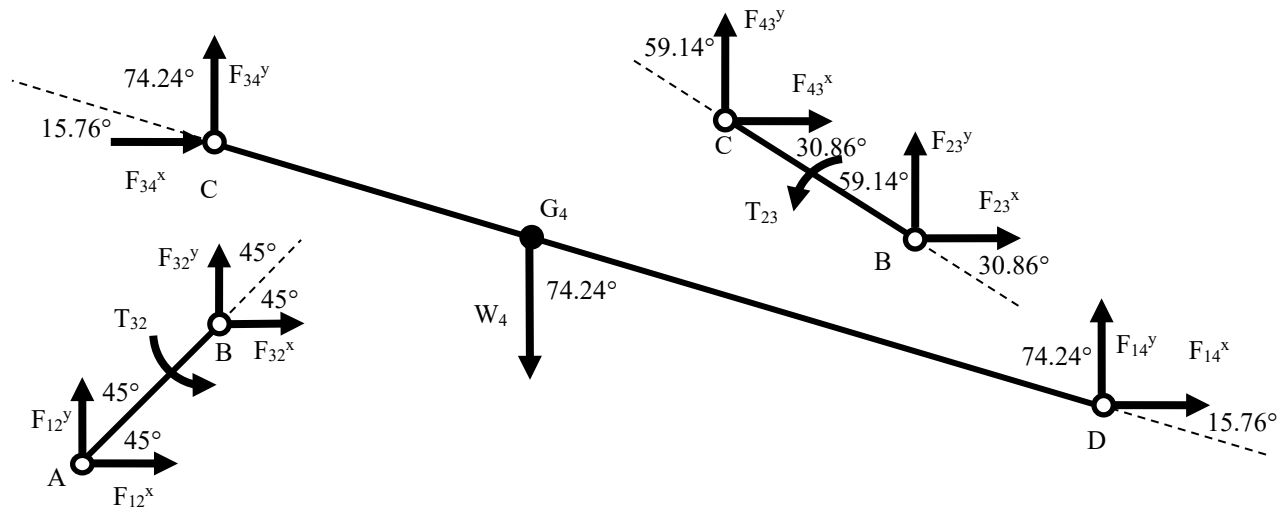
Do the magnitude and direction for your answer seem reasonable? Why?

direction seems OK approximate solution $W_4 \text{ (DG}_4) \sim F_{34}^y \text{ (CD)}, F_{34}^y \sim 122 \text{ lbf} \sim F_{12}^y$
 $F_{12}^y \text{ (AB)} \sim T_{32} \sim 1464 \text{ in.lbf}$

Rate the last four assumptions and state your reasoning.

- constant ω_4 1=poor 2=acceptable for an approximation 3=very good
- weight of arms is negligible 1=poor 2=acceptable for an approximation 3=very good
- friction is negligible 1=poor 2=acceptable for an approximation 3=very good
- no muscle force at A, C, D 1=poor 2=acceptable for an approximation 3=very good

Determine ω_4 of the torso/legs when the forearm is aligned with the upper arms ($\theta_2 = \theta_3$). $\omega_4 = 0$



ΣF on 2 right +

$$F_{12}^x + F_{32}^x = 0$$

ΣF on 2 up +

$$F_{12}^y + F_{32}^y = 0$$

ΣM on 2 about A CCW +

$$-(F_{32}^x \sin 45^\circ) AB + (F_{32}^y \sin 45^\circ) AB + T_{32} = 0$$

ΣF on 3 right +

$$F_{23}^x + F_{43}^x = 0$$

ΣF on 3 up +

$$F_{23}^y + F_{43}^y = 0$$

ΣM on 3 about B CCW +

$$-(F_{43}^x \sin 30.86^\circ) BC - (F_{43}^y \sin 59.14^\circ) BC + T_{23} = 0$$

ΣF on 4 right +

$$F_{34}^x + F_{14}^x = 0$$

ΣF on 4 up +

$$F_{34}^y + F_{14}^y - W_4 = 0$$

ΣM on 4 about D CCW +

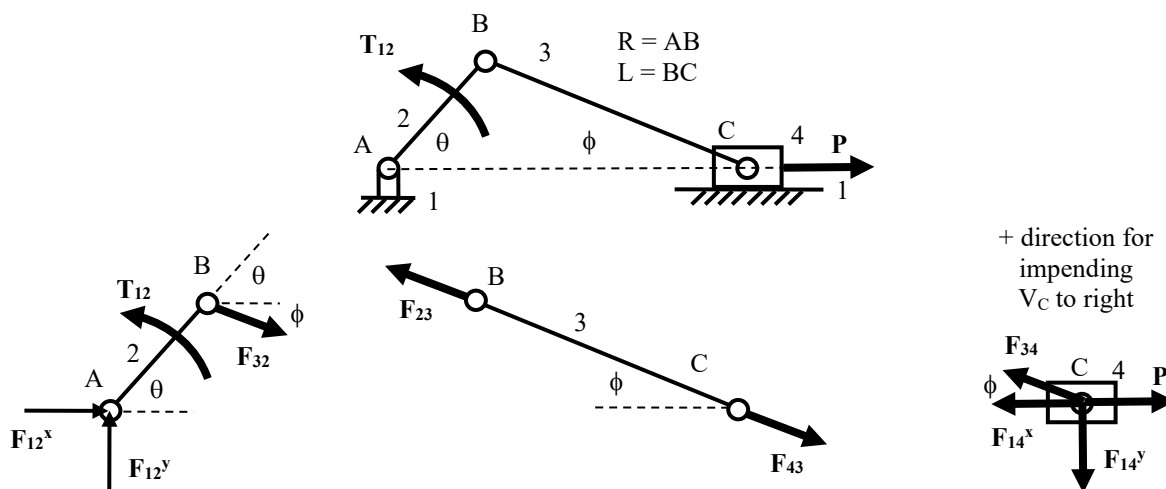
$$-(F_{34}^x \sin 15.76^\circ) CD - (F_{34}^y \sin 74.24^\circ) CD + (W_4 \sin 74.24^\circ) DG_4 = 0$$

$$\begin{bmatrix}
 1 & 0 & -1 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 1 & 0 & -1 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & AB\sin 45^\circ & -AB\sin 45^\circ & 0 & 0 & 0 & 0 & 1 \\
 0 & 0 & 1 & 0 & -1 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 1 & 0 & -1 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & BC\sin 30.86^\circ & BC\sin 59.14^\circ & 0 & 0 & -1 \\
 0 & 0 & 0 & 0 & 1 & 0 & 1 & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & 1 & 0 & 1 & 0 \\
 0 & 0 & 0 & 0 & -CD\sin 15.76^\circ & -CD\sin 74.24^\circ & 0 & 0 & 0
 \end{bmatrix}
 \begin{Bmatrix}
 F_{12}^x \\
 F_{12}^y \\
 F_{23}^x \\
 F_{23}^y \\
 F_{34}^x \\
 F_{34}^y \\
 F_{14}^x \\
 F_{14}^y \\
 T_{32}
 \end{Bmatrix}
 =
 \begin{Bmatrix}
 0 \\
 0 \\
 0 \\
 0 \\
 0 \\
 0 \\
 0 \\
 + W_4 \\
 -(W_4 \sin 74.24^\circ)DG_4
 \end{Bmatrix}$$

$$\begin{bmatrix}
 1 & 0 & -1 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 1 & 0 & -1 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 8.4853 & -8.4853 & 0 & 0 & 0 & 0 & 1 \\
 0 & 0 & 1 & 0 & -1 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 1 & 0 & -1 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & 7.1812 & 12.0179 & 0 & 0 & -1 \\
 0 & 0 & 0 & 0 & 1 & 0 & 1 & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & 1 & 0 & 1 & 0 \\
 0 & 0 & 0 & 0 & -15.6718 & -55.5309 & 0 & 0 & 0
 \end{bmatrix}
 \begin{Bmatrix}
 F_{12}^x \\
 F_{12}^y \\
 F_{23}^x \\
 F_{23}^y \\
 F_{34}^x \\
 F_{34}^y \\
 F_{14}^x \\
 F_{14}^y \\
 T_{32}
 \end{Bmatrix}
 =
 \begin{Bmatrix}
 0 \\
 0 \\
 0 \\
 0 \\
 0 \\
 0 \\
 0 \\
 180 \\
 -6756.1
 \end{Bmatrix}$$

$$\begin{Bmatrix}
 F_{12}^x \\
 F_{12}^y \\
 F_{23}^x \\
 F_{23}^y \\
 F_{34}^x \\
 F_{34}^y \\
 F_{14}^x \\
 F_{14}^y \\
 T_{32}
 \end{Bmatrix}
 =
 \begin{Bmatrix}
 -29.3 \text{ lbf} \\
 129.9 \text{ lbf} \\
 -29.3 \text{ lbf} \\
 129.9 \text{ lbf} \\
 -29.3 \text{ lbf} \\
 129.9 \text{ lbf} \\
 29.3 \text{ lbf} \\
 50.1 \text{ lbf} \\
 1351.1 \text{ in.lbf}
 \end{Bmatrix}$$

Static Force Analysis for Slider Crank - Scalar



$$\Sigma F \text{ on 4 right} + \quad + P - F_{14}^x \text{ sign}(V_C) - F_{34} \cos \phi = 0$$

$$\Sigma F \text{ on 4 up} + \quad + F_{34} \sin \phi - F_{14}^y = 0$$

$$\Sigma M \text{ on 3} \quad F_{23} = F_{43}$$

$$\Sigma M \text{ on 2 about A CCW} + \quad + T_{12} - (F_{32} \sin(\theta + \phi)) R = 0$$

$$\text{friction} \quad F_{14}^x = \mu F_{14}^y$$

$$F_{34} = \frac{P}{(\cos \phi + \text{sign}(V_C) \mu \sin \phi)}$$

$$F_{14}^y = \frac{P \sin \phi}{(\cos \phi + \text{sign}(V_C) \mu \sin \phi)} = P \tan \phi \text{ for } \mu = 0$$

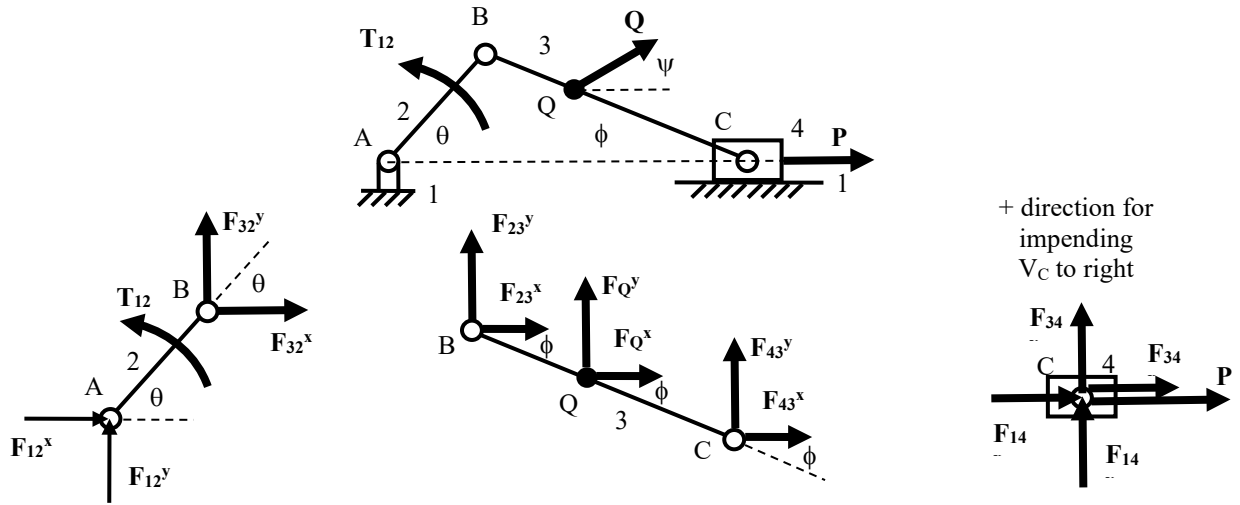
$$F_{14}^x = \frac{\mu P \sin \phi}{(\cos \phi + \text{sign}(V_C) \mu \sin \phi)}$$

$$F_{12}^x = -\frac{P \cos \phi}{(\cos \phi + \text{sign}(V_C) \mu \sin \phi)} = -P \text{ for } \mu = 0$$

$$F_{12}^y = \frac{P \sin \phi}{(\cos \phi + \text{sign}(V_C) \mu \sin \phi)} = P \tan \phi \text{ for } \mu = 0$$

$$T_{12} = \frac{P R \sin(\theta + \phi)}{(\cos \phi + \text{sign}(V_C) \mu \sin \phi)}$$

Static Force Analysis for Slider Crank - Matrix



$$\begin{aligned}
 \Sigma F \text{ on 2 right} + & F_{12}^x + F_{32}^x = 0 \\
 \Sigma F \text{ on 2 up} + & F_{12}^y + F_{32}^y = 0 \\
 \Sigma M \text{ on 2 about A CCW} + & -(F_{32}^x \sin \theta) AB + (F_{32}^y \cos \theta) AB + T_{12} = 0 \\
 \\
 \Sigma F \text{ on 3 right} + & F_{23}^x + F_{43}^x + F_Q^x = 0 \\
 \Sigma F \text{ on 3 up} + & F_{23}^y + F_{43}^y + F_Q^y = 0 \\
 \Sigma M \text{ on 3 about Q CCW} + & -(F_{23}^x \sin \phi) BQ - (F_{23}^y \cos \phi) BQ + (F_{43}^x \sin \phi) CQ + (F_{43}^y \cos \phi) CQ = 0 \\
 \\
 \Sigma F \text{ on 4 right} + & F_{14}^x + F_{34}^x + P = 0 \\
 \Sigma F \text{ on 4 up} + & F_{14}^y + F_{34}^y = 0 \\
 \text{friction} & F_{14}^x = -\mu \text{ abs}(F_{14}^y) \text{ sign}(V_C)
 \end{aligned}$$

$$\begin{bmatrix}
 1 & 0 & -1 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 1 & 0 & -1 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & +AB \sin \theta & -AB \cos \theta & 0 & 0 & 0 & 0 & 1 \\
 0 & 0 & 1 & 0 & -1 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 1 & 0 & -1 & 0 & 0 & 0 \\
 0 & 0 & -BQ \sin \phi & -BQ \cos \phi & +CQ \sin \phi & +CQ \cos \phi & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & 1 & 0 & 1 & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & 1 & 0 & 1 & 0 \\
 0 & 0 & 0 & 0 & 0 & 0 & 1 & +\mu \text{ sign}(V_C) & 0
 \end{bmatrix}
 \begin{Bmatrix}
 F_{12}^x \\
 F_{12}^y \\
 F_{23}^x \\
 F_{23}^y \\
 F_{34}^x \\
 F_{34}^y \\
 F_{14}^x \\
 F_{14}^y \\
 T_{12}
 \end{Bmatrix}
 =
 \begin{Bmatrix}
 0 \\
 0 \\
 0 \\
 -F_Q^x \\
 -F_Q^y \\
 0 \\
 -P \\
 0 \\
 0
 \end{Bmatrix}$$

