

1) Plot the crank torque required to drive the web cutter at constant crank speed 60 rpm CCW through one full revolution of the crank. Assume that the mechanism operates in a vertical plane, that there are no external forces on the mechanism other than the weight of the links, and that friction is negligible. Neglect cutting forces.

constant speed crank 6.28 rad/sec CCW

2) What are the bearing reactions and crank torque for part 1) when $\phi_2 = 85.5$ degrees?

$$T_{1on2} \underline{0.377 \text{ N.cm CW}} \quad F_{1on2} (\underline{0.113}, \underline{0.391})^T \text{ N} \quad F_{2on3} (\underline{0.115}, \underline{0.113})^T \text{ N}$$

$$F_{3on4} (\underline{0.351}, \underline{-1.338})^T \text{ N} \quad F_{1on4} (\underline{-0.543}, \underline{2.820})^T \text{ N}$$

3) Assume that the crank of the web cutter is rotating at constant crank speed 60 rpm CCW when the crank motor power is suddenly cut at $\phi_2 = 85.5^\circ$. Determine the acceleration of the crank at the moment when crank torque is cut to zero, and predict crank angle and crank speed at $h = 0.05$ seconds later.

$$\text{use } \{q\}_{\text{NEW}} = \{q\}_{\text{OLD}} + \{\dot{q}\}_{\text{OLD}} h + 0.5\{\ddot{q}\}_{\text{OLD}} h^2 \quad \text{and} \quad \{\dot{q}\}_{\text{NEW}} = \{\dot{q}\}_{\text{OLD}} + \{\ddot{q}\}_{\text{OLD}} h$$

$$\ddot{\phi}_2 \underline{21.72 \text{ rad/sec}^2 \text{ CCW}} \quad \dot{\phi}_2 \text{ at } h \text{ later } \underline{7.37 \text{ rad/sec CCW}} \quad \phi_2 \text{ at } h \text{ later } \underline{105.06 \text{ deg}}$$

4) Check the kinematic consistency of your $\{q\}_{\text{NEW}}$ and $\{\dot{q}\}_{\text{NEW}}$ predicted at h later.

$$\max \text{ abs } \{\Phi\} \underline{0.0177 \text{ cm}} \quad \max \text{ abs } [\Phi_q] \{\dot{q}\} \underline{1.0490 \text{ cm/sec}}$$

EXTRA CREDIT

Determine the STATIC crank torque and bearing reactions at $\phi_2 = 85.5^\circ$ required to produce 10 N vertical cutting force between cutters P and Q of the web cutter. Assume there are no external forces on the mechanism other than the vertical cutting forces. Assume that friction is negligible. Do not include the weight of the links. Do not include dynamic effects.

$$T_{1on2} \underline{31.56 \text{ N.cm CCW}} \quad F_{1on2} (\underline{-7.64}, \underline{3.54})^T \text{ N} \quad F_{2on3} (\underline{-7.64}, \underline{3.54})^T \text{ N}$$

$$F_{3on4} (\underline{-7.64}, \underline{13.54})^T \text{ N} \quad F_{1on4} (\underline{+7.64}, \underline{-3.54})^T \text{ N}$$

EXTRA EXTRA CREDIT

Use velocities and virtual work to determine STATIC crank torque at $\phi_2 = 85.5^\circ$ per above.

$$T_{1on2} \underline{31.57 \text{ N.cm CCW}}$$

Parts 1 and 2 - INVERSE DYNAMICS

$$\{Q_{on 2}\} = \begin{Bmatrix} 0 \\ -m_2 g \\ 0 \end{Bmatrix} \quad \{Q_{on 3}\} = \begin{Bmatrix} 0 \\ -m_3 g \\ 0 \end{Bmatrix} \quad \{Q_{on 4}\} = \begin{Bmatrix} 0 \\ -m_4 g \\ 0 \end{Bmatrix} \quad \{Q\}_{APPLIED} = \begin{Bmatrix} \{Q_{on 2}\} \\ \{Q_{on 3}\} \\ \{Q_{on 4}\} \end{Bmatrix}$$

$$[M]_{9 \times 9} \{\ddot{q}\}_{9 \times 1} + [\Phi_q]_{9 \times 9}^T \{\lambda\}_{9 \times 1} = \{Q\}_{APPLIED} \quad \{\lambda\} = ([\Phi_q]_{9 \times 9}^T)^{-1} (\{Q\}_{APPLIED} - [M]_{9 \times 9} \{\ddot{q}\})$$

$$\begin{bmatrix} [M]_{9 \times 9} & [\Phi_q]_{9 \times 9}^T \\ [\Phi_q]_{9 \times 9} & [0]_{9 \times 9} \end{bmatrix}_{18 \times 18} \begin{Bmatrix} \{\ddot{q}\}_{9 \times 1} \\ \{\lambda\}_{9 \times 1} \end{Bmatrix}_{18 \times 1} = \begin{Bmatrix} \{Q\}_{APPLIED} \\ \{\gamma\} \end{Bmatrix}_{18 \times 1}$$

$$\text{units} \quad m = [\text{kg}] \quad \ddot{r} = [\text{cm}/\text{sec}^2] \quad J' = [\text{kg} \cdot \text{cm}^2] \quad \ddot{\phi} = [\text{rad}/\text{sec}^2] \\ F = [\text{kg} \cdot \text{cm}/\text{sec}^2] \quad T = [\text{kg} \cdot \text{cm}^2/\text{sec}^2]$$

$$\{\Phi\}_{REV} = \{r_j\}^P - \{r_i\}^P \quad \{\lambda\}_{REV} = -\{F_{on j}\} = -\{F_{i on j}\}$$

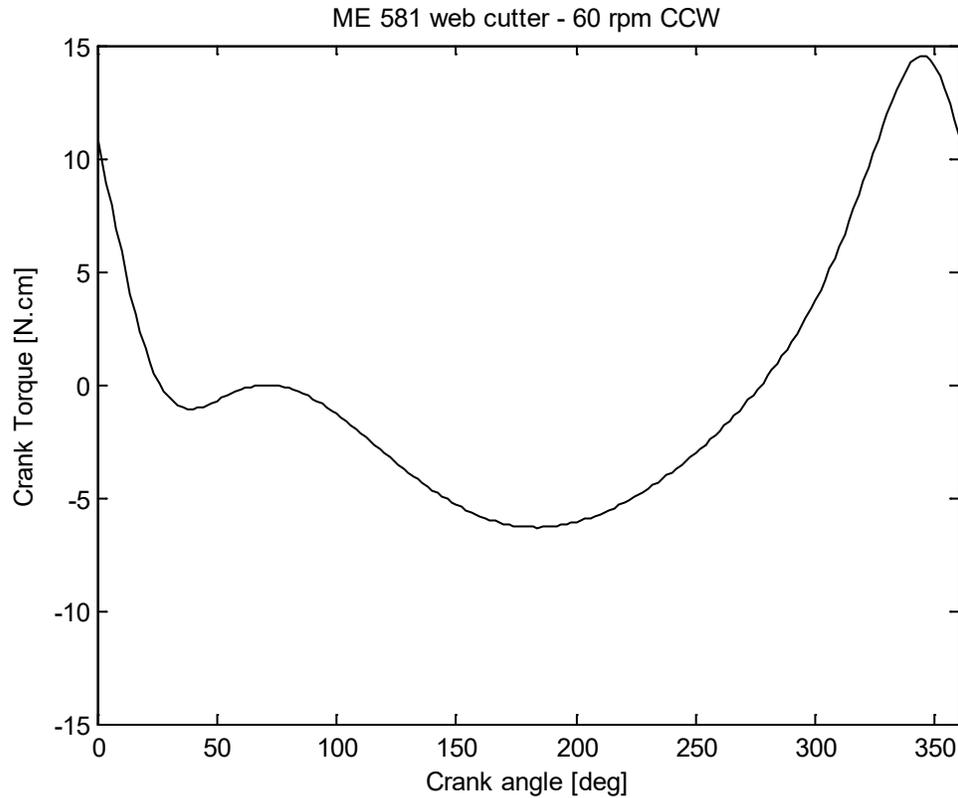
$$\Phi_{DRIVER} = \phi_j - \phi_i - C - f(t) \quad \lambda_{DRIVER} = -T_{on j}$$

$$\{\Phi\} = \begin{Bmatrix} \{r_2\}^A - \{r_1\}^A \\ \{r_3\}^B - \{r_2\}^B \\ \{r_4\}^C - \{r_3\}^C \\ \{r_4\}^D - \{r_1\}^D \\ \phi_2 - \phi_{2_0} - \omega_2 t \end{Bmatrix} \quad \begin{matrix} j=2, i=1 \\ j=3, i=2 \\ j=4, i=3 \\ j=4, i=1 \\ j=2, i=1 \end{matrix} \quad \{\lambda\} = \begin{Bmatrix} -\{F_{1 on 2}\} \\ -\{F_{2 on 3}\} \\ -\{F_{3 on 4}\} \\ -\{F_{1 on 4}\} \\ -T_{1 on 2} \end{Bmatrix}$$

Same code for

`we_ini_c03.m` `we_phi.m` `we_kin.m`

Verified same graphs as C03



```
% wc_main_c04_part1.m - web cutter four-bar for ME 581
% main for C04 part 1
% HJSIII - 20.03.04
```

```
clear
```

```
% general constants
```

```
d2r = pi / 180;
R = [ 0 -1; 1 0 ];
```

```
% initialize - centroidal coordinates
```

```
wc_ini_c03
```

```
% mass matrix - units [kg] and [kg*cm*cm]
```

```
m2 = 0.03082;
m3 = 0.15962;
m4 = 0.15180;
J2p = 0.091;
J3p = 6.448;
J4p = 6.808;
```

```
M = diag( [ m2 m2 J2p m3 m3 J3p m4 m4 J4p ] );
```

```
% applied external forces - units [kg*cm/s/s]
```

```
% weight due to gravity in negative y direction
```

```
Q2=[ 0 -981*m2 0 ]';
Q3=[ 0 -981*m3 0 ]';
Q4=[ 0 -981*m4 0 ]';
QA=[ Q2 ; Q3 ; Q4 ];
```

```
% starting position
```

```
phi2_start = 0 * d2r; % start at zero
%phi2_start = 30 * d2r; % position in line drawing
%phi2_start = 90 * d2r; % check at 90 degrees
%phi2_start = 85.5 * d2r; % cutter closing
%q = [ 0 0 4.4925 -0.8726 -3.9037 1.7233 -3.0341 10.1612 -0.6438]'; % q at opening
```

```

%phi2_start = 257.4 * d2r; % cutter opening

% time loop
%tpr = 0; % single poosition
tpr = 2 * pi / w2; % one revolution at constant speed
t_start = 0; % start
t_end = tpr; % end
nt = 180; % number of time steps
dt = (t_end - t_start) / nt;

keep_q = [];
keep = [];
for t = t_start : dt : t_end;

% kinematics
wc_kin

% inverse dynamics - solve for Lagrange multipliers
lambda = inv(JAC') * ( QA - M*qdd );

% revolute bearing forces - units [kg*cm/s/s]
% constraints = rj-ri for r2A-r1A, r3B-r2B, r4C-r3C, r4D-r1D, phi2-phi1
% lambda = -F i on j
F1on2 = -lambda(1:2) / 100; % convert to N, j=2 i=1
F2on3 = -lambda(3:4) / 100; % convert to N, j=3 i=2
F3on4 = -lambda(5:6) / 100; % convert to N, j=4 i=3
F1on4 = -lambda(7:8) / 100; % convert to N, j=4 i=1
T1on2 = -lambda(9) / 100; % convert to N.cm, j=2 i=1

% check crank torque by simple virtual work
T1on2_vw = -qd' * ( QA - M*qdd ) / phi2d;
T1on2_vw = T1on2_vw / 100; % convert to N.cm

% save kinematics and forces
ang2 = phi2 / d2r;
detJAC = det(JAC);
keep_q = [ keep_q ; q' qd' qdd' ];
keep = [ keep ; detJAC T1on2 T1on2_vw ];

% bottom - for t
end

% plot
ang2 = keep_q(:,3) /d2r;
phi3dd = keep_q(:,24);
phi4dd = keep_q(:,27);
detJAC = keep(:,1);
T1on2 = keep(:,2);
T1on2_vw = keep(:,3);

figure( 1 )
plot( ang2,phi3dd,'k--', ang2,phi4dd,'k-' )
axis( [ 0 360 -40 40 ] )
title( 'ME 581 web cutter - 60 rpm CCW' )
xlabel( 'Crank angle [deg]' )
ylabel( 'Angular acceleration [rad/s/s]' )
legend( 'Link 3', 'Link 4' )

figure( 2 )
plot( ang2,detJAC,'k' )
axis( [ 0 360 -300 0 ] )
title( 'ME 581 web cutter - 60 rpm CCW' )
xlabel( 'Crank angle [deg]' )
ylabel( 'Determinant of Jacobian [cm^2]' )

figure( 3 )
plot( ang2,T1on2,'k' )
axis( [ 0 360 -15 15 ] )
title( 'ME 581 web cutter - 60 rpm CCW' )
xlabel( 'Crank angle [deg]' )
ylabel( 'Crank Torque [N.cm]' )

figure( 4 )

```

```
plot( ang2,T1on2,'k', ang2,T1on2_vw,'ro' )
axis( [ 0 360 -15 15 ] )
title( 'ME 581 web cutter - 60 rpm CCW' )
xlabel( 'Crank angle [deg]' )
ylabel( 'Crank Torque [N.cm]' )

% bottom - wc_main_c04_part1
```

```

+++++
>> wc_main_c04_part2

Flon2_x   Flon2_y   F2on3_x   F2on3_y   F3on4_x   F3on4_y   Flon4_x   Flon4_y   Tlon2
   [N]     [N]       [N]       [N]       [N]       [N]       [N]       [N]       [N.cm]
0.1133    0.3905    0.1152    0.1125    0.3512    -1.3378   -0.5429    2.8204   -0.3767

Tlon2_vw = -0.3767

+++++

% wc_main_c04_part2.m - web cutter four-bar for ME 581
%   main for C04 part 2
% HJSIII - 20.03.04

clear

% general constants
d2r = pi / 180;
R = [ 0 -1; 1 0 ];

% initialize - centroidal coordinates
wc_ini_c03

% mass matrix - units [kg] and [kg*cm*cm]
m2 = 0.03082;
m3 = 0.15962;
m4 = 0.15180;
J2p = 0.091;
J3p = 6.448;
J4p = 6.808;

M = diag( [ m2 m2 J2p m3 m3 J3p m4 m4 J4p ] );

% applied external forces - units [kg*cm/s/s]
% weight due to gravity in negative y direction
Q2=[ 0 -981*m2 0 ]';
Q3=[ 0 -981*m3 0 ]';
Q4=[ 0 -981*m4 0 ]';
QA=[ Q2 ; Q3 ; Q4 ];

% starting position
phi2_start = 85.5 * d2r; % start when cutter is closing
t = 0;

% kinematics
wc_kin

% inverse dynamics - solve for Lagrange multipliers
lambda = inv(JAC') * ( QA - M*qdd );

% revolute bearing forces - units [kg*cm/s/s]
% constraints = rj-ri for r2A-r1A, r3B-r2B, r4C-r3C, r4D-r1D, phi2-phi1
% lambda = -F i on j
Flon2 = -lambda(1:2) / 100; % convert to N, j=2 i=1
F2on3 = -lambda(3:4) / 100; % convert to N, j=3 i=2
F3on4 = -lambda(5:6) / 100; % convert to N, j=4 i=3
Flon4 = -lambda(7:8) / 100; % convert to N, j=4 i=1
Tlon2 = -lambda(9) / 100; % convert to N.cm, j=2 i=1

% check crank torque by simple virtual work
Tlon2_vw = -qd' * ( QA - M*qdd ) / phi2d;
Tlon2_vw = Tlon2_vw / 100; % convert to N.cm

% show results
disp( '   Flon2_x   Flon2_y   F2on3_x   F2on3_y   F3on4_x   F3on4_y   Flon4_x   Flon4_y   Tlon2' )
disp( '           [N]     [N]       [N]       [N]       [N]       [N]       [N]       [N]       [N.cm]' )
disp( [ Flon2' F2on3' F3on4' Flon4' Tlon2 ] )

Tlon2_vw

% bottom - wc_main_c04_part2

```

Parts 3 and 4 - FORWARD DYNAMICS

remove driver constraint

remove row 9 of $\{\Phi\}$ $[\Phi_q]$ $\{v\}$ $\{\gamma\}$ $\{\lambda\}$

$$\begin{bmatrix} [M]_{9 \times 9} & [\Phi_q]^T_{9 \times 8} \\ [\Phi_q]_{8 \times 9} & [0]_{8 \times 8} \end{bmatrix}_{17 \times 17} \begin{Bmatrix} \{\ddot{q}\}_{9 \times 1} \\ \{\lambda\}_{8 \times 1} \end{Bmatrix}_{17 \times 1} = \begin{Bmatrix} \{Q\}_{APPLIED}_{9 \times 1} \\ \{\gamma\}_{8 \times 1} \end{Bmatrix}_{17 \times 1}$$

+++++

>> wc_main_c04_part3

phi2dd = 21.7197 phi2d_new = 7.3692 phi2_new = 105.0556

PHI_test =
-0.0118 0.0177 -0.0159 0.0120 0.0050 -0.0076 -0.0002 0.0070vel_test =
-0.7702 1.0490 -1.0025 0.7281 0.3209 -0.4652 -0.0164 0.4515

+++++

% wc_main_c04_part3.m - web cutter four-bar for ME 581
% main for C04 part 3
% HJSIII - 20.03.04

clear

% general constants

d2r = pi / 180;
R = [0 -1; 1 0];

% initialize - centroidal coordinates

wc_ini_c03

% mass matrix - units [kg] and [kg*cm*cm]

m2 = 0.03082;
m3 = 0.15962;
m4 = 0.15180;
J2p = 0.091;
J3p = 6.448;
J4p = 6.808;

M = diag([m2 m2 J2p m3 m3 J3p m4 m4 J4p]);

% applied external forces - units [kg*cm/s/s]
% weight due to gravity in negative y direction
Q2=[0 -981*m2 0]';
Q3=[0 -981*m3 0]';
Q4=[0 -981*m4 0]';
QA=[Q2 ; Q3 ; Q4];

% starting position

phi2_start = 85.5 * d2r; % start when cutter is closing
t = 0;

% standard kinematics immediately before driver is cut

wc_kin

% forward dynamics - solve for qdd and Lagrange multipliers simultaneously

% NOTE: torque on link2 is cut to zero

JAC89 = JAC(1:8,1:9);

```
gamma8 = accrhs(1:8);
EOMmat = [ M      JAC89'      ;
           JAC89  zeros(8,8) ];
EOMrhs = [ QA      ;
           gamma8 ];
EOMSol = inv(EOMmat) * EOMrhs;
qdd = EOMSol(1:9);
lambda = EOMSol(10:17);

% predict one time step forward
h = 0.05;
q_new = q + qd*h + qdd*h*h/2;
qd_new = qd + qdd*h;

phi2dd = qdd(3)
phi2d_new = qd_new(3)
phi2_new = q_new(3) /d2r

% check kinematic position consistency at predicted time step
% NOTE: this will change JAC just a little
q = q_new;
wc_phi
PHI_test = PHI(1:8)'

% check kinematic position consistency at predicted time step
vel_test = ( JAC(1:8,:) * qd_new )'

% bottom - wc_main_c04_part3
```

Extra credit - **STATIC**

$$\{F\}^P = \begin{Bmatrix} 0 \\ +10 \text{ N} \end{Bmatrix} \quad \{F\}^Q = \begin{Bmatrix} 0 \\ -10 \text{ N} \end{Bmatrix}$$

$$\{Q_{on2}\} = \begin{Bmatrix} 0 \\ 0 \\ 0 \end{Bmatrix} \quad \{Q_{on3}\} = \begin{Bmatrix} \{F\}^P \\ \{s_3\}^{pT} [B_3]^T \{F\}^P \end{Bmatrix} \quad \{Q_{on4}\} = \begin{Bmatrix} \{F\}^Q \\ \{s_4\}^{qT} [B_4]^T \{F\}^Q \end{Bmatrix}$$

$$\{Q\}_{APPLIED} = \begin{Bmatrix} \{Q_{on2}\} \\ \{Q_{on3}\} \\ \{Q_{on4}\} \end{Bmatrix}$$

$$[M]\{\ddot{q}\} + [\Phi_q]^T \{\lambda\} = \{Q\}_{APPLIED} \quad \{\ddot{q}\} = \{0\} \quad \{\lambda\} = ([\Phi_q]^T)^{-1} \{Q\}_{APPLIED}$$

VIRTUAL WORK

$$T_{12} \circ \omega_2 + \{F\}^P \circ \{\dot{r}_3\}^P + \{F\}^Q \circ \{\dot{r}_4\}^Q = 0$$

units $\omega = [\text{rad/sec}]$ $F = [\text{N}]$ $\dot{r} = [\text{cm/sec}]$ $T = [\text{N.cm}]$

```

++++
>> wc_main_c04_ec

      Flon2_x   Flon2_y   F2on3_x   F2on3_y   F3on4_x   F3on4_y   Flon4_x   Flon4_y   T1on2
      [N]       [N]       [N]       [N]       [N]       [N]       [N]       [N]       [N.cm]
      -7.6376   3.5386   -7.6376   3.5386   -7.6376   13.5386   7.6376   -3.5386   31.5666

T12_vw = 31.5666

++++

% wc_main_c04_ec.m - web cutter four-bar for ME 581
%   main for C04 extra credit
% HJSIII - 20.03.04

clear

% general constants
d2r = pi / 180;
R = [ 0 -1; 1 0 ];

% initialize - centroidal coordinates
wc_ini_c03

% starting position
phi2_start = 85.5 * d2r; % start when cutter is closing
t = 0;

% kinematics - need position, need Jacobian
%   use velocity for virtual work, do not need acceleration
wc_kin

% applied external forces - units [N]

```

```

% 10 N vertical cutter forces between points P3 and Q4
FP = [ 0  10 ]';
FQ = [ 0 -10 ]';

Q2 = [ 0 0 0 ]';
Q3 = [ FP      ;
      s3pP'*B3'*FP ];
Q4 = [ FQ      ;
      s4pQ'*B4'*FQ ];
QA = [ Q2 ; Q3 ; Q4 ];

% inverse dynamics - solve for Lagrange multipliers
lambda = inv(JAC') * QA;

% revolute bearing forces - units [kg*cm/s/s]
% constraints = rj-ri for r2A-r1A, r3B-r2B, r4C-r3C, r4D-r1D, phi2-phi1
% lambda = -F i on j
F1on2 = -lambda(1:2);      % j=2 i=1  already [N], do not convert
F2on3 = -lambda(3:4);      % j=3 i=2
F3on4 = -lambda(5:6);      % j=4 i=3
F1on4 = -lambda(7:8);      % j=4 i=1
T1on2 = -lambda(9);        % j=2 i=1

% show results
disp( '   F1on2_x  F1on2_y  F2on3_x  F2on3_y  F3on4_x  F3on4_y  F1on4_x  F1on4_y
T1on2' )
disp( '           [N]           [N]           [N]           [N]           [N]           [N]           [N]           [N]
[N.cm]' )
disp( [ F1on2'  F2on3'  F3on4'  F1on4'  T1on2 ] )

% check crank torque by simple virtual work
T12_vw = -( FP'*r3Pd + FQ'*r4Qd ) / phi2d

% bottom - wc_main_c04_ec

```